Selecting Optimal Heat Transfer Chloride Salt for Molten Salt Fast Reactor: Heat Exchanger Design and Mass Comparison

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1. Introduction

In the pursuit of sustainable and efficient nuclear energy solutions, Molten Salt Fast Reactors (MSFRs) have emerged as promising candidates, distinguished by their inherent safety features and enhanced efficiency. A critical aspect influencing the performance of MSFRs lies in the choice of the heat transfer fluid. The selection of an optimal chloride salt needs to consider various aspects of the salt, such as thermal properties, heat exchanger design, and pumping work consumption. In previous studies, the thermodynamic properties and heat exchanger design of NaCl-MgCl2, KCl-MgCl2, and NaCl-KCl-ZnCl₂ were compared [1,2]. A Plate-Fin Heat Exchanger was selected for the intermediate heat exchangers based on the literature review. However, for the sake of simplicity and the limited amount of data, the fuel salt was assumed to have the same thermal properties as the heat transfer salt. This study introduces PFHE design considerations with NaCl-UCl₃ as the fuel salt, accompanied by an initial mass evaluation for each heat exchanger core.

2. Methods and Results

2.1 Thermal Sizing

Table I. Air Brayton Cycle Information [2]

Input Parameter			
Thermal output	8.0 MW _{th}		
Air temperature	15°C		
Turbine inlet temperature	630 °C		
Turbine efficiency	88 %		
Compressor efficiency	84 %		
Recuperator effectiveness	90 %		
Result			
Mass Flow Rate	35.70 kg/s		
Thermal efficiency	31.05 %		

Table II. Mass Flow Rate of each candidate & fuel salt [2]

	Mass Flow Rate [kg/s]
NaCl-MgCl ₂	147.61
KCl-MgCl ₂	138.05
NaCl-KCl-ZnCl ₂	174.48
NaCl-UCl ₃	271.19

Table I shows the Air Brayton cycle parameters, and Table II lists the mass flow rates for each candidate and fuel salt, derived from preceding research [2]. Based on the mass flow rate of the heat transfer salt, the mass flow rate of the fuel salt is calculated before designing the primary heat exchanger. The eutectic point of NaCl-UCl₃ (65.9%-34.1%, mol %) is selected as the fuel salt in this research. The thermal properties of the fuel salt are obtained from the literatures [3-5]. The density of NaCl-UCl₃, not directly available, is deduced using the additive molar volume method. The properties used in this study are given below. Based on the fuel salt mass flow rate and thermal properties, the primary heat exchangers are designed.

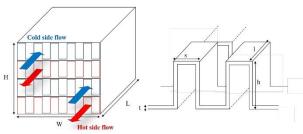
$$\rho\left[\frac{kg}{m^3}\right] = 4254.7 - 1.0621T[K] \tag{1}$$

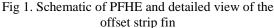
$$C_p \left[\frac{J}{kg - K} \right] = 590 \tag{2}$$

$$\lambda \left[\frac{W}{m - K} \right] = 0.0012T[K] - 0.495$$
(3)

$$\eta[Pa * s] = 0.1207 \exp(-0.0067T[K])$$
(4)
+ 0.0037 exp(-0.001T[K])

2.2 PFHE Design





As shown in Figure 1, counter flow PFHEs with offset strip fins are used as the primary and secondary intermediate heat exchangers for this paper. This study's methodology mirrors previous works, with NaCl-UCl₃ now introduced as the fuel salt [2]. Thus, PFHE core designs for secondary heat exchangers were the same. Table III presents the PFHE design result, leading to an estimation of each PFHE core's mass.

		-	2	e		
	Primary PFHE (With NaCl-UCl ₃ as fuel salt)		Secondary PFHE (With air Brayton cycle)			
	NaCl-MgCl ₂	KCl-MgCl ₂	NaCl-KCl- ZnCl ₂	NaCl-MgCl ₂	KCl-MgCl ₂	NaCl-KCl- ZnCl ₂
Hot side inlet/outlet temperature [°C]	650 / 600		640 / 590			
Cold side inlet/outlet temperature [°C]	590 / 640		426.09 / 630			
Plate Thickness [m]	$5.0 imes 10^{-4}$					
HX width [m]	0.75	0.75	1.5	1.8	1.8	1.8
HX length [m]	0.979	0.975	0.661	0.769	0.765	0.804
HX height [m]	0.546	0.546	0.770	0.629	0.629	0.658
Hot side pressure drop [kPa]	136.58	136.16	137.549	5.995	6.515	87.923
Cold side pressure drop [kPa]	80.10	83.79	193.137	8.424	8.379	9.3674
Hot Side Mass Flow Rate [kg/s]		271.19		147.612	138.054	174.482
Cold Side Mass Flow Rate [kg/s]	147.612	138.054	174.482		35.70	

Table III. Primary and Secondary PFHE Core Design Result

Table IV. Primary and Secondary PFHE Core Mass Estimation

	Primary PFHE (With NaCl-UCl ₃ as fuel salt)		Secondary PFHE (With air Brayton cycle)			
	NaCl-MgCl ₂	KCl-MgCl ₂	NaCl-KCl- ZnCl ₂	NaCl-MgCl ₂	KCl-MgCl ₂	NaCl-KCl- ZnCl ₂
Hot Side Fluid Volume [m ³]	0.126	0.124	0.238	0.166	0.166	0.166
Cold Side Fluid Volume [m ³]	0.127	0.125	0.239	0.484	0.484	0.484
Hot side Fluid Mass [kg]	412.238	407.187	779.710	329.937	324.480	376.102
Cold Side Fluid Mass [kg]	253.037	247.053	498.378	0.592	0.589	0.647
Structure Mass [kg]	1687.100	1666.428	2937.027	2271.944	2259.660	2473.824
Total Core Mass [kg]	2352.375	2320.668	4215.115	2602.473	2584.728	2850.572

2.3 Mass Calculation

Before calculating the PFHE core mass, the thick ness of the end plate and side plate need to be determined. The thickness is calculated based on ASME BPVC section VIII, Division 1.

$$t = \frac{P * R}{SE - 0.6P} \tag{5}$$

Hastelloy N is used as the material for the PFHE, because Hastelloy N has high corrosive resistance in molten salt environment. The maximum allowable stress of the material is obtained from the literature review [6]. Assuming the joint efficiency is 1 and factor of safety 3.5, the thickness of the heat exchangers is calculated to be 1.5 cm.

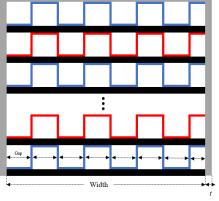


Fig 2. Frontal cross-section view of PFHE Core

Figure 2 offers a frontal cross-sectional view of a PFHE core, with Table IV summarizing the mass calculation results. The mass of the fluids and the structure are calculated using the density of the fluid and

the material. As shown in the result, the calculation reveals a negligible mass difference between the PFHEs employing NaCl-MgCl₂ and KCl-MgCl₂. However, the NaCl-KCl-ZnCl₂-based PFHE needs a larger heat transfer area, owing to its inferior thermal properties, especially volumetric heat capacity, thus increasing the overall mass.

3. Summary and Conclusions

In this research, the selection of the optimal heat transfer chloride salt for MSFR with focus on heat exchanger design with NaCl-UCl₃ as the fuel salt and mass calculation are conducted. The use of NaCl-UCl₃ as the primary PFHE's hot side fluid unveils the significant impact of varying heat transfer salts on design parameters. Furthermore, the study meticulously calculates the mass and volume occupied by the fluids, adhering to ASME BPVC standards for structural components. The results of this study suggest KCl-MgCl₂ as the superior heat transfer salt, considering heat exchanger volume and mass constraints. In this paper, only the core part of the heat exchanger is included. For the further work, the design and the mass calculation of the header and fluid distributor should further proceed to compare options in more detail.

Symbol [Unit]	Definition
$\boldsymbol{\rho} \left[\frac{\mathrm{kg}}{\mathrm{m}^3} \right]$	Density
$C_p \left[\frac{J}{kg - K} \right]$	Heat Capacity
$\lambda \left[\frac{W}{m-K}\right]$	Thermal Conductivity
$\boldsymbol{\eta} [Pa * s]$	Dynamic Viscosity
t [m]	Thickness
P [Pa]	Design Pressure
R [m]	Equivalent Radius
S [Pa]	Maximum Allowable Stress
E [-]	Joint Efficiency

NOMENCLATURE

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