

Preliminary design study of S-CO₂ power conversion system for PG-SFR

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1. Introduction

PG-SFR, which stands for Prototype Generation IV - Sodium-cooled Fast Reactor is currently under development mainly by KAERI. The original design adopted a conventional steam Rankine cycle for the power conversion system but violent Sodium Water Reaction (SWR) raises a concern on the system safety. To suggest an alternative option, previous studies explored the SFR application of various gas Brayton

cycles to increase the safety and simplify the safety system [1, 2, 3].

A previous study [4] compared and evaluated the performance of S-CO₂, helium and nitrogen Brayton cycles for the SFR power conversion systems. Among those Brayton cycles, S-CO₂ cycle can achieve high efficiency even comparable with the steam Rankine cycle [4]. This paper discusses the preliminary S-CO₂ cycle design for PG-SFR condition considering the size and efficiency.

S-CO ₂ cycle design condition				
Compressor Outlet Pressure [MPa]	20	State	T [°C]	P [MPa]
Turbine Inlet Temperature [°C]	505	Turbine in	505	19.9
Turbine efficiency [%]	90	Turbine out	394.6	7.62
Main Compressor / Recompressing Compressor efficiency [%]	80	High Temperature Recuperator hotside out	141.6	7.56
Recompressing fraction [%]	27	Precooler in	65.3	7.52
HTR / LTR effectiveness [%]	95	Main Compressor in	31.3	7.5
HTR hot / cold pressure drop [kPa]	60 / 30	Main Compressor out	62.1	20
LTR hot / cold pressure drop [kPa]	40 / 20	Low Temperature Recuperator coldside out	119.1	19.98
PC [CO ₂] pressure drop [kPa]	20	Recompressing Compressor out	159.1	19.98
IHX [CO ₂] pressure drop [kPa]	50	High Temperature Recuperator coldside in	128.9	19.98
Thermal efficiency [%]	40.66	Intermediate heat exchanger in	337.5	19.95

Table I. Design variables of S-CO₂ cycle

	A	B	C
Heat [MWth / Unit]	368.9	184.45	122.97
Electric Power [MWe / Unit]	150	75	50
CO ₂ mass flow [kg/s]	1792.9	896.4	597.6

Table II. Design options of S-CO₂ cycle

2. S-CO₂ cycle design

Even though sodium reacts more mildly with CO₂ compared to water, the Na-CO₂ reaction is exothermic and must be controlled for the reactor safety. To ensure the safety, several options of S-CO₂ power conversion systems are considered. As Table I and II shows, three PCS options (single, double and triple PCS) and two TM options (single shaft design and triple shaft design) can be considered and analyzed in terms of size and total CO₂ inventory.

2.1 Turbomachinery design

The preliminary study for the turbomachinery is performed by considering specific speed (n_s) and specific diameter (d_s) of turbomachineries [5]. For a Brayton cycle application, many designs adopt single shaft design in which all turbomachineries are connected to a single shaft. As compressors operate with the mechanical work delivered from turbines, the motor requirements are reduced. To overcome the rotating speed limitation while the motor requirements are minimized, Triple Shaft Design (TSD) is suggested by Lee [6]. In this design, the electricity is generated by the power turbine and two additional turbines provide mechanical work to two compressors. TSD is advantageous because all turbomachineries can be designed with a single stage which can be beneficial

both for operation and maintenance. These two designs are compared in Table III.

2.2 Heat exchanger design

Major studies and researches have been conducted on Printed Circuit Heat Exchanger (PCHE) for the S-CO₂ cycle application. PCHE has a wide operational range up to 900°C and 40 MPa while the high compactness can be achieved as well [7]. KAIST-HXD is constructed by KAIST research team to design and evaluate the PCHE performance. KAIST-HXD utilizes

the experimental correlation based on the open literature data and NGO correlation for Nusselt number is applied for the turbulent regime up to 22,000 Reynolds number [8]. Future studies on a correlation applicable to higher Reynolds number are required.

For the intermediate heat exchanger design, the sodium properties are referred from the previous research work [9]. Hejzlar recommended a general correlation for the liquid metal heat transfer and friction factor [10]. The total heat exchanger volume and CO₂ mass for each case are shown in Table II.

	150 MWe					75 MWe					50 MWe				
TM-design	SSD														
component	RPM	D (mm)	efficiency	RPM	D (mm)	efficiency	RPM	D (mm)	efficiency	RPM	D (mm)	efficiency	RPM	D (mm)	efficiency
turbine	3,600	1,830	>90	7,200	929	>85	10,800	586	>90						
MC	3,600	940	>70	7,200	478	>80	10,800	321	>80						
RC	3,600	1,218	>65	7,200	583	>70	10,800	398	>75						
TM-design	TSD														
Component	RPM	D (mm)	efficiency	RPM	D (mm)	efficiency	RPM	D (mm)	efficiency	RPM	D (mm)	efficiency	RPM	D (mm)	efficiency
power turbine	3,600	1,536	>85	7,200	700	>90	10,800	505	>90						
MC turbine	6,600	671	>80	9,400	478	>80	11,500	390	>75						
RC turbine	7,000	678	>70	10,000	476	>70	11,000	414	>75						
MC	6,600	517	>80	9,400	341	>80	11,500	279	>80						
RC	7,000	798	>70	10,000	558	>70	11,000	509	>75						
HX-design	IHX	HTR	LTR	PC	total	IHX	HTR	LTR	PC	total	IHX	HTR	LTR	PC	total
effectiveness %	NA	95.7	95.1	NA		NA	95.1	95.1	NA		NA	95.3	95.1	NA	
Hot channel pressure drop kPa	8.6	52.8	36.8	4.8	150.6	8.6	32.1	23.5	4.8	112.0	8.6	57.8	22.3	4.8	144.1
Cold channel pressure drop kPa	25.3	16.7	14.0	174.4		25.4	10.7	15.4	174.4		25.4	19.1	14.7	174.4	
Volume m ³	2.3	13.8	18.7	2.6	37.4	1.1	7.3	9.3	1.3	19.1	0.8	4.4	6.4	0.9	12.4
Length m	0.3	0.8	1.3	0.3	NA	0.3	0.7	1.1	0.3	NA	0.3	0.8	1.1	0.3	NA
CO ₂ mass kg	107	1049	2605	228	3988	53	555	1114	114	1836 (3872)	36	331	763	76	1205 (3615)
Pumping power kW	88.4	NA	NA	928	1016	44.2	NA	NA	463.9	508 (1016)	29.5	NA	NA	309.3	339 (1017)

Table III. S-CO₂ cycle module component design variables

3. Summary and future works

For the power conversion system of PG-SFR, 150, 75, 50MWe S-CO₂ cycle modules were designed considering turbomachinery for multiple cycle layouts. In a single shaft design, all turbomachineries are connected to a single shaft, which the motor requirements are reduced but the rotating speed is limited at the same time. On the other hand, a power turbine and compressor turbines are connected in a different shaft which rotating speed is less limited and all turbomachineries can be designed with a single stage in a triple shaft design. Further studies for the Double Shaft Design (DSD) in which the main and recompressing compressors and turbines are connected to a single shaft while only power turbine is separated will be performed in the future.

For the heat exchanger volume and CO₂ mass analysis, the total mass does not increase linearly as the power conversion system size increases. As the number of module increases, the CO₂ mass in S-CO₂ cycle decreases. In addition, as number of modules increases, less CO₂ leaks in a system when the system fails. Future studies on the sodium and CO₂ reaction will be necessary to ensure the safety of S-CO₂ power conversion system coupled to PG-SFR.

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