

# Interfacial Friction Factor Model for Two-dimensional Film Flow

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# 1. INTRODUCTION

2. 2D Film Flow Experiments

3. Momentum Conservation Equations

4. Mechanistic Model for Interfacial Friction Factor

5. Conclusions

## ❖ Trend of thermal hydraulic analysis in nuclear engineering

- Multi-scale, multi-dimensional, multi-phase, multi-physics ...

→ HIGH PRECISION ANALYSIS!!

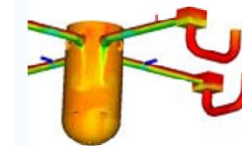
- Two-phase flow analysis



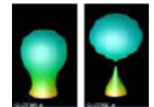
System Scale



Component Scale



CFD Scale



DNS Scale



## V&V Multi-dimensional modules

- RELAP-5
- MARS
- TRACE
- CATHARE3
- SPACE



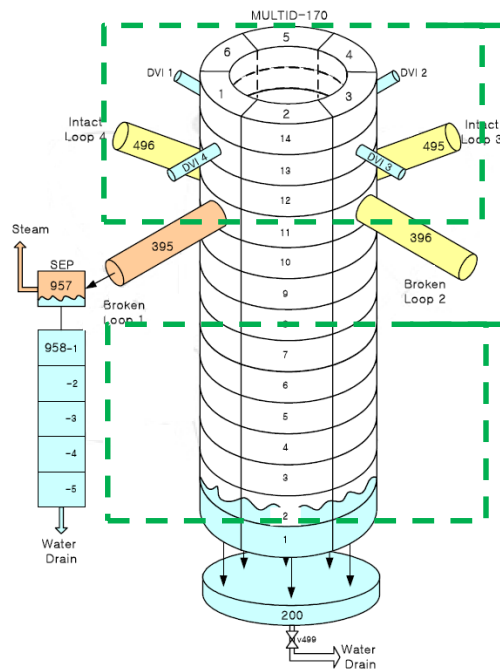
## Computational Multi-Fluid Dynamics

- Neptune-CFD
- STAR-CCM
- CUPID

## ❖ Motivation of this study

- To produce benchmark data for multidimensional codes validation.
- To develop constitutive models based on [multidimensional local measurements](#).

## ➡ Verification & Validation (V&V) of multidimensional codes



Upper downcomer

- ① MIDAS (KAERI)

Lower downcomer

- ① ROCOM (Rossendorf)
- ② DOBO (KAERI)
- ③ DYNAS (KAERI)

## Challenges...

- ✓ Local measurement data in upper downcomer were insufficient.
- ✓ 1D empirical models have been used.

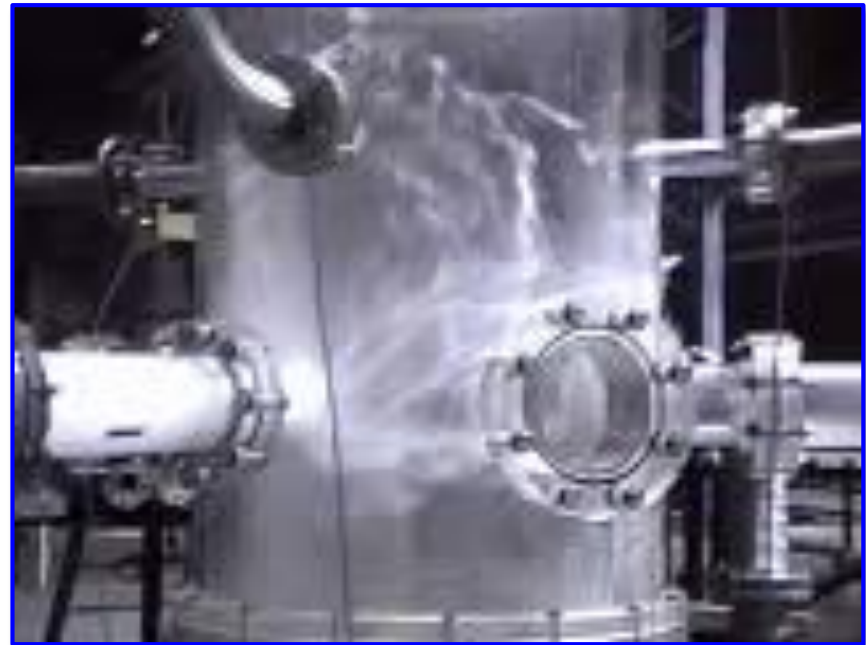
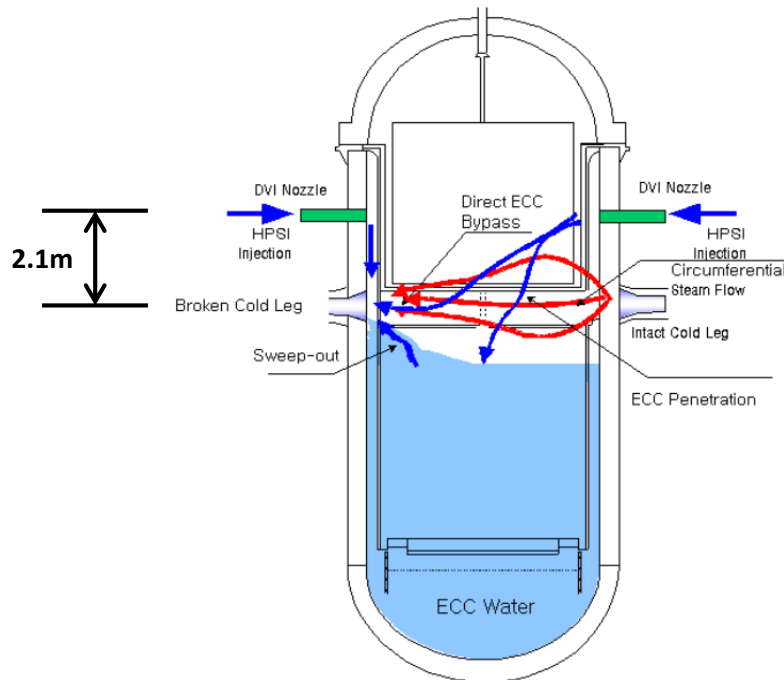
Multidimensional Codes Analysis

**nuTHEL**

Nuclear Thermal Hydraulic Engineering Lab.

## ❖ Upper downcomer multidimensional two-phase flow

- ECC(Emergency Core Coolant) bypass occurs during reflood phase of cold-leg LBLOCA
- 2.1m elevated DVI(Direct Vessel Injection) line in APR1400
- 2-fluid, 2-D film flow: Vertical liquid flow vs Circumferential vapor flow



## ❖ Objectives

**2D Local Experimental Data Acquisition in Downcomer**



**Liquid Film Experiment with Lateral Air Flow**

Liquid film velocity

Liquid film thickness

Air velocity



**2D Film Flow Interfacial Friction Factor**



**Validation of Multidimensional Code**

## 1. Introduction

# 2. 2D FILM FLOW EXPERIMENTS

## 3. Momentum Conservation Equations

## 4. Mechanistic Model for Interfacial Friction Factor

## 5. Conclusions

# 2D Film Flow Experiments

## ❖ Experimental Features

- 1/10 scaled down facility of unfolded downcomer
- Local liquid film velocity & thickness measurement
- Test conditions

	Real conditions	Scaled-down conditions*
Nozzle diameter (m)	0.22	<b>0.022</b>
Water inlet velocity (m/s)	2	<b>0.63</b>
Lateral air velocity (m/s)	15 ~ 45	<b>5 ~ 15</b>



### \*Modified linear scaling

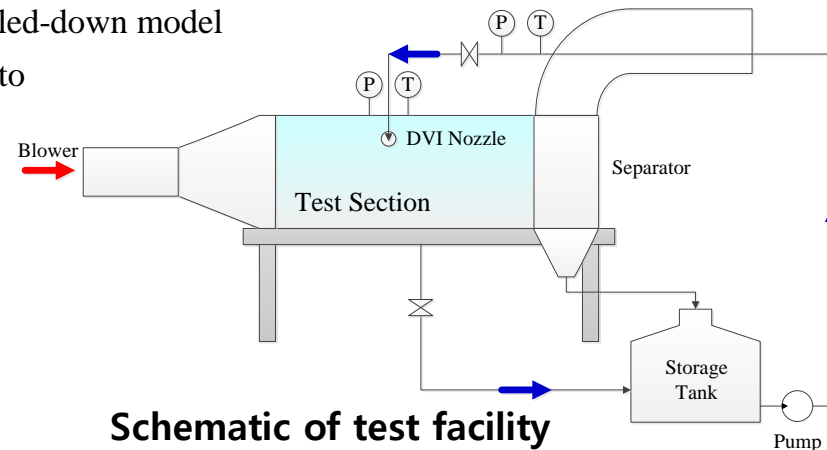
(B.J. Yun et al, "Scaling for the ECC bypass phenomena during the LBLOCA reflood phase," NED, 2004)

$$j_k^* = \frac{\dot{m}_k}{\rho_k A_{flow}} \left[ \frac{\rho_k}{(\rho_f - \rho_g) g D_{gap}} \right]^{1/2} \Rightarrow V_m = \frac{V_p}{\sqrt{S}}$$

$V_m$  : velocity of scaled-down model

$V_p$  : velocity of proto

$S$  : scaling ratio





# Flow Shape Change with Lateral Air Velocity

## ❖ Hydrodynamic condition

- Inlet air velocity: 5 ~ 15 m/s (real scale 15 ~ 45m/s)

0 m/s



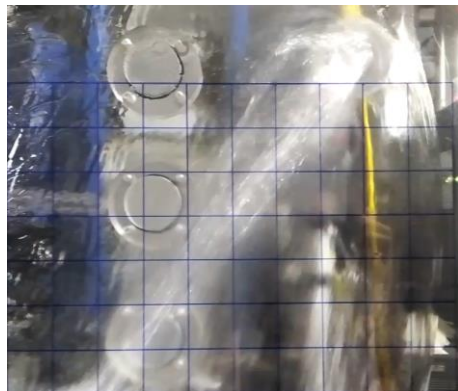
5 m/s



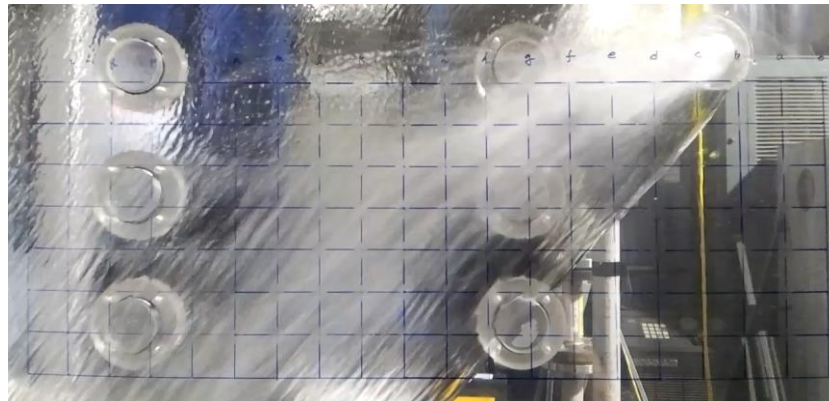
7 m/s



9 m/s

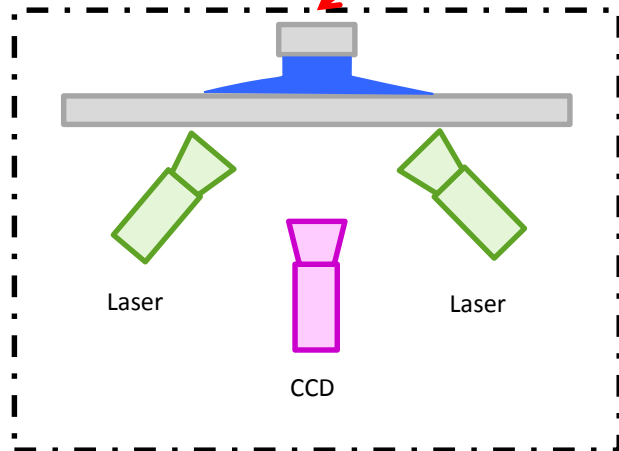
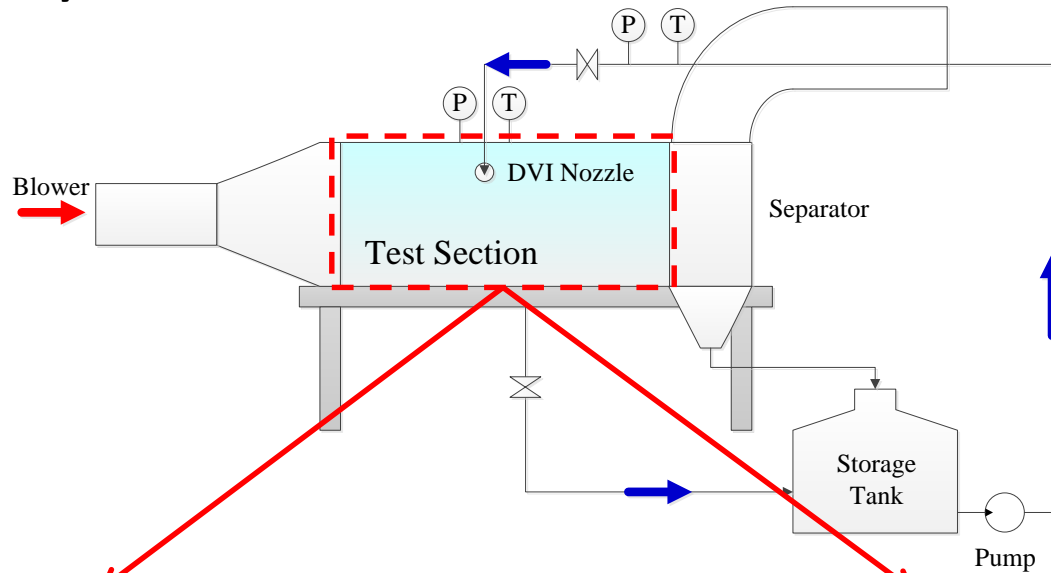


15 m/s



# Experimental Measurement Methods

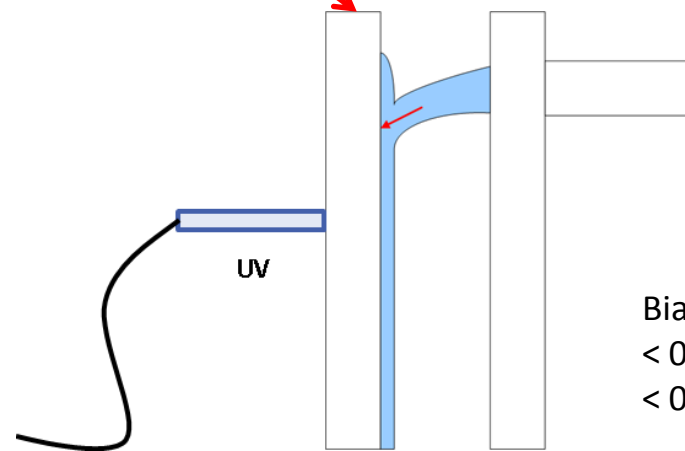
## ❖ Liquid film velocity and thickness



Uncertainty  
5.65%

Liquid film velocity measurement  
with **volume-averaged PIV\*** method

\*Particle Image Velocimetry

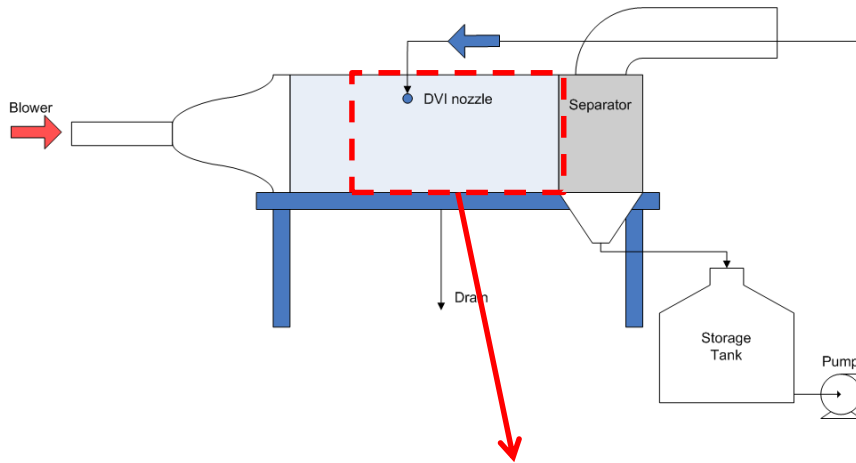


Bias error  
< 0.2 % (Temp.)  
< 0.04 mm (Time)

Liquid film thickness measurement  
with **ultrasonic thickness gauge**

# Liquid Film Velocity Measurement

## ❖ Control volume for liquid film velocity measurement



- 560nm long-pass filter
- Pulse: 500  $\mu$ s
- Sampling: 100 Hz
- 500 pairs (5s)
- Adaptive PIV
  - Min. 32 pixels
  - Max. 64 Pixels

30 x 30 mm

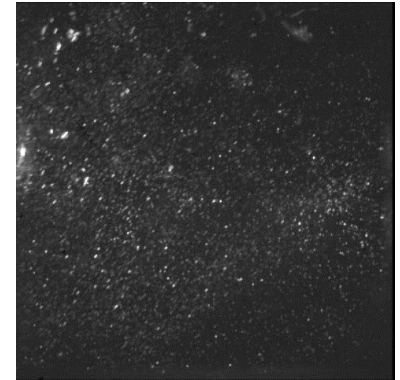
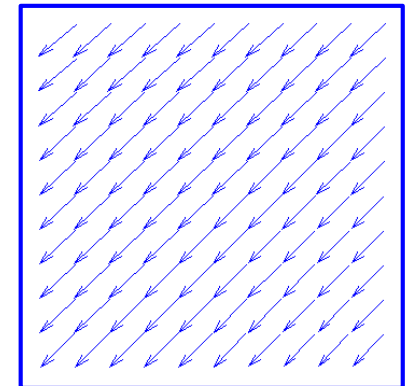
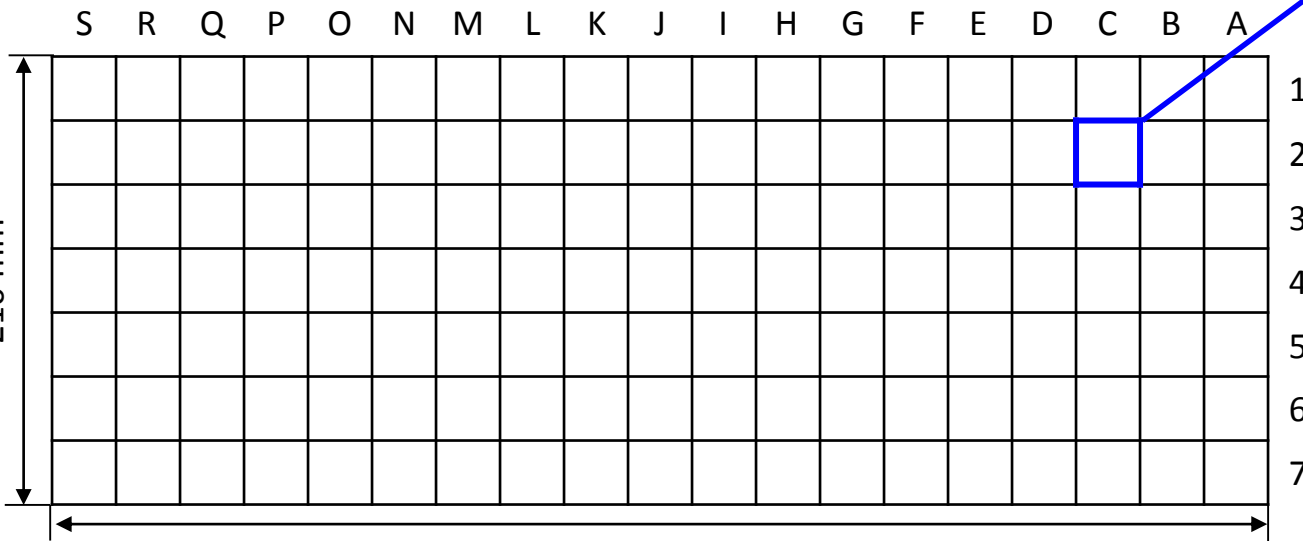


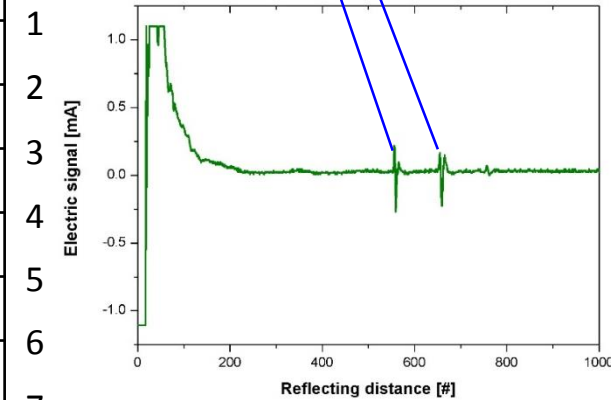
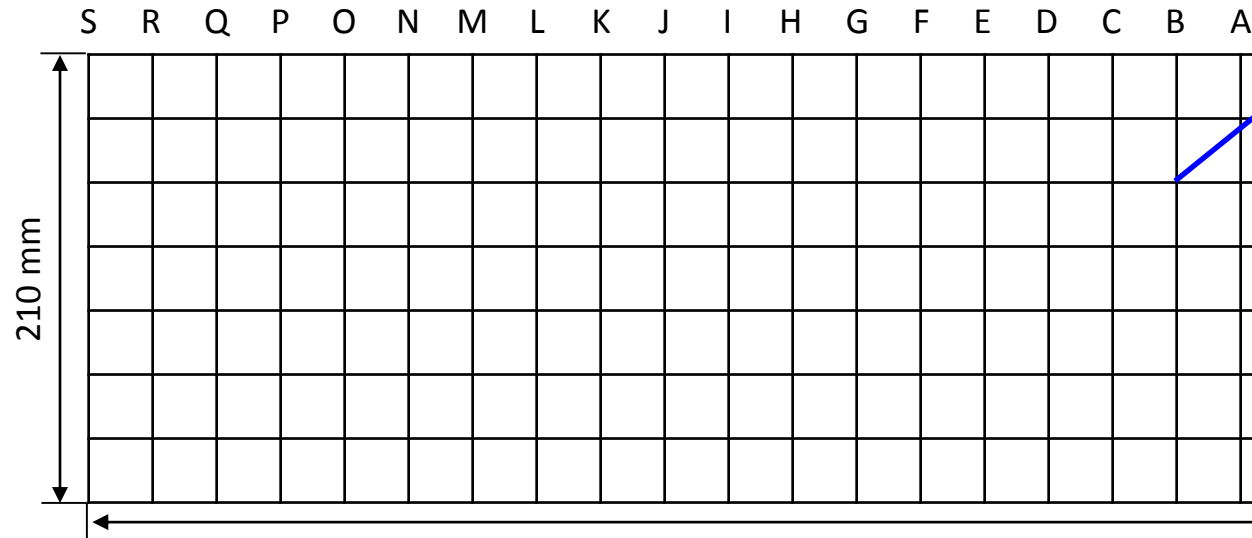
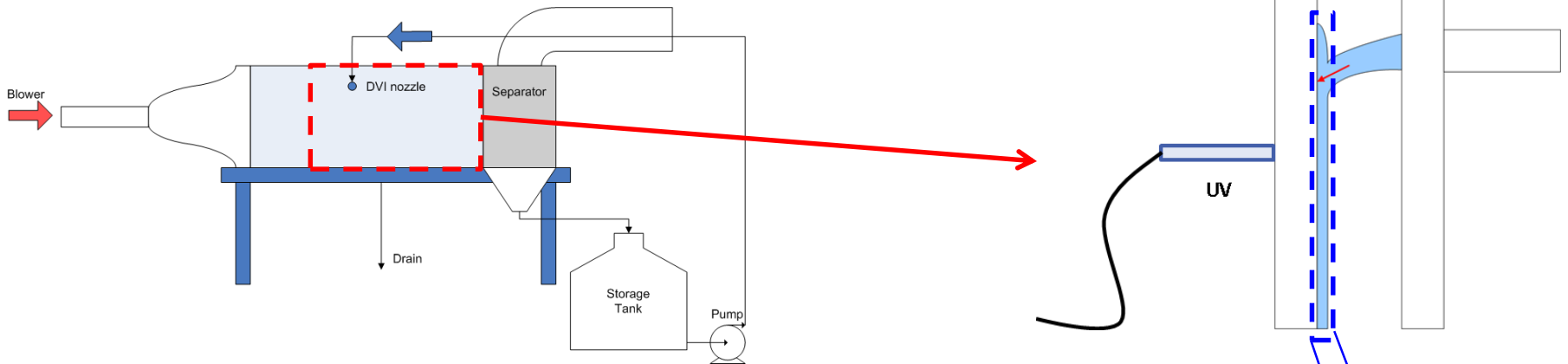
Image Pairs



Analysis result

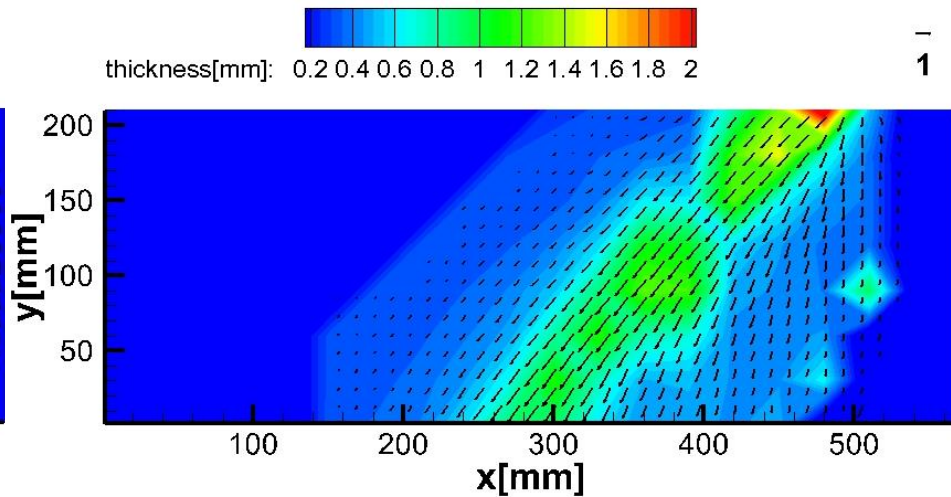
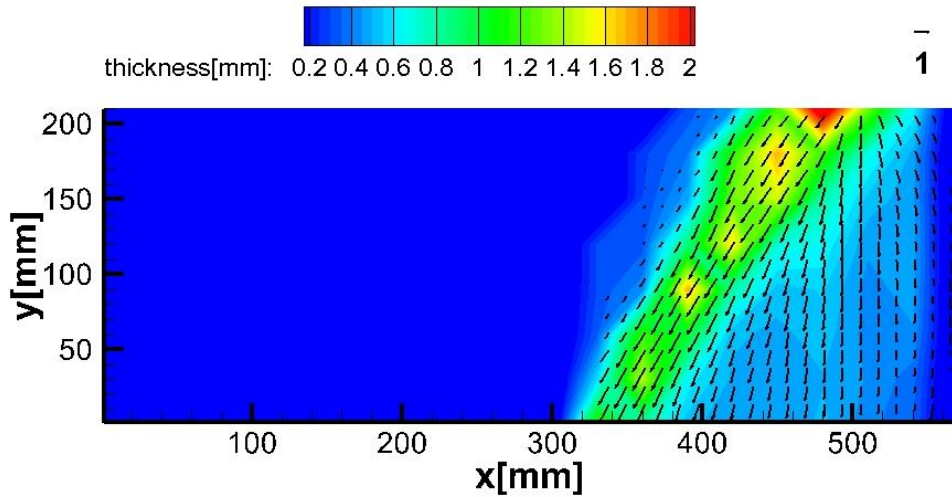
# Liquid Film Thickness Measurement

## ❖ Measurement points of liquid film thickness

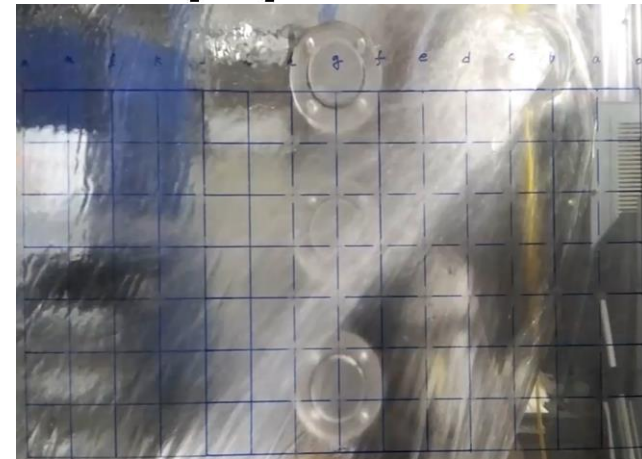


# Results of 2D Film Flow Experiments

## ❖ Results: 9, 11, 13, 15 m/s lateral air injection (1/2)



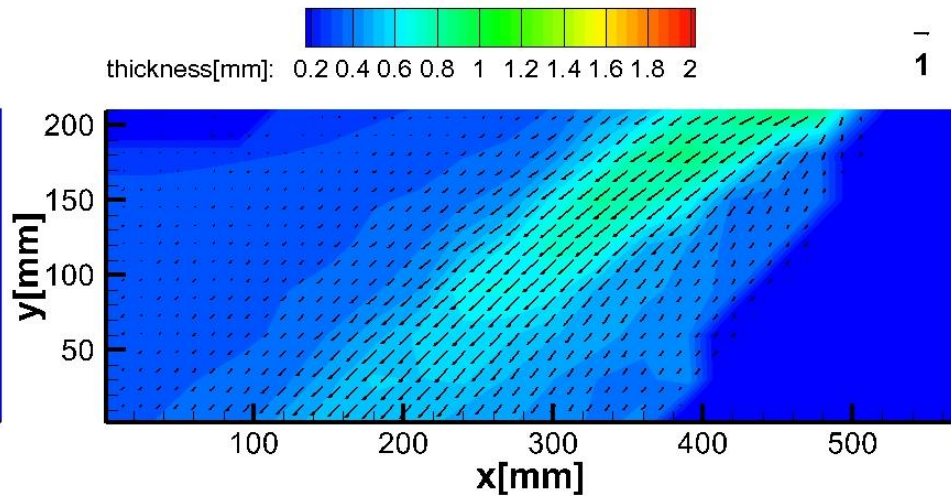
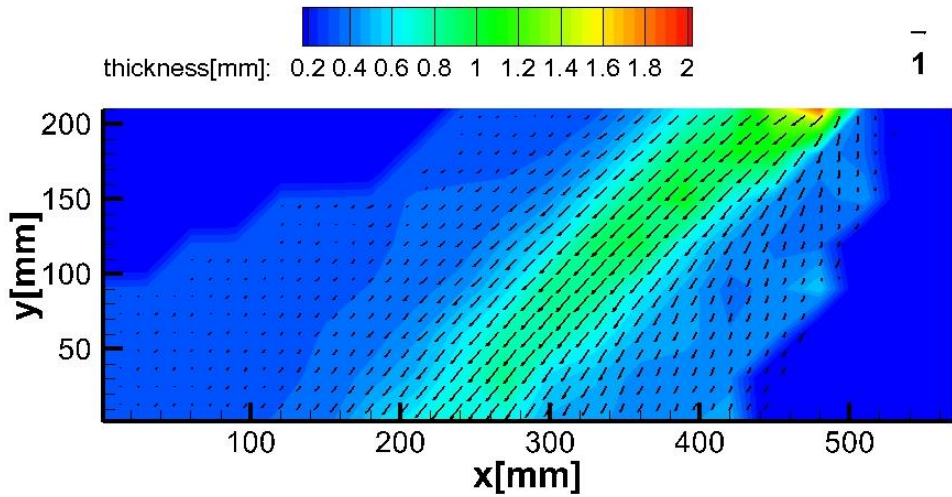
9 m/s



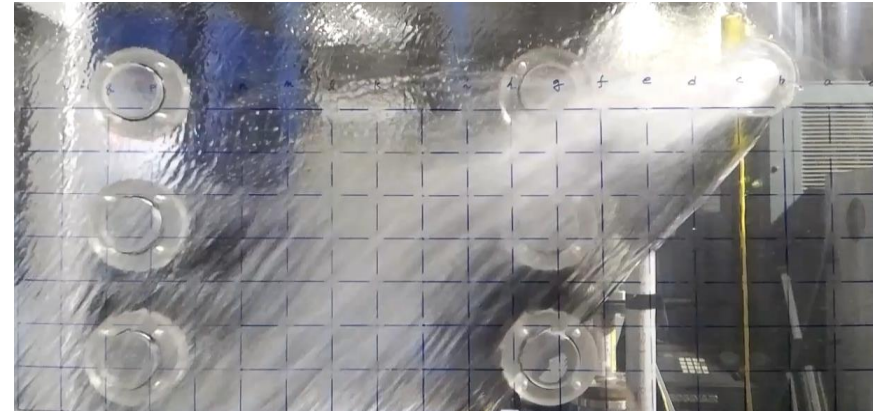
11 m/s

# Results of 2D Film Flow Experiments

## Results: 9, 11, 13, 15 m/s lateral air injection (2/2)



13 m/s



15 m/s

1. Introduction

2. 2D Film Flow Experiments

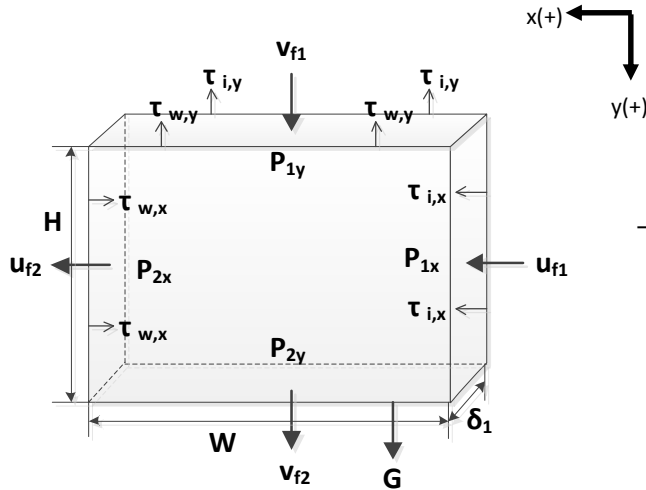
## **3. MOMENTUM CONSERVATION EQUATIONS**

4. Mechanistic Model for Interfacial Friction Factor

5. Conclusions

# Momentum Conservation Equations

## Force balance of liquid film

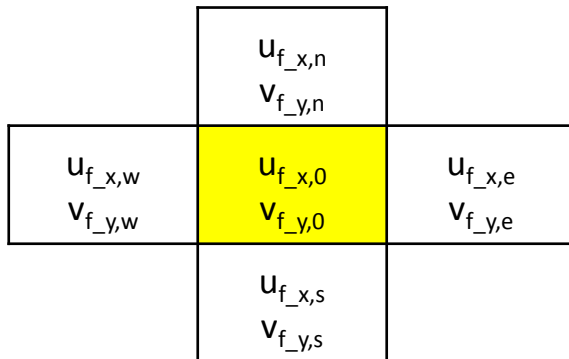


X-direction

$$\begin{aligned}
 & -(P_2 - P_1)_x A_1 + \int_A \tau_{i,x} dA_1 - \int_A \tau_{wf,x} dA_1 \\
 & = \sum_{in\_x} (\rho_f A_1 U_{f\_face} \times U_n) - \sum_{out\_x} (\rho_f A_1 U_{f\_face} \times U_n)
 \end{aligned}$$

Y-direction

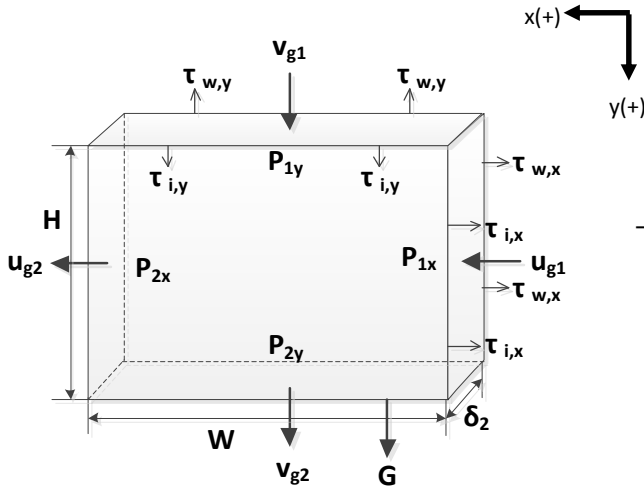
$$\begin{aligned}
 & -(P_2 - P_1)_y A_1 - \int_A \tau_{i,y} dA_1 - \int_A \tau_{wf,y} dA_1 + \rho_f gV \\
 & = \sum_{in\_y} (\rho_f A_1 U_{f\_face} \times U_n) - \sum_{out\_y} (\rho_f A_1 U_{f\_face} \times U_n)
 \end{aligned}$$





# Momentum Conservation Equations

## Force balance of air

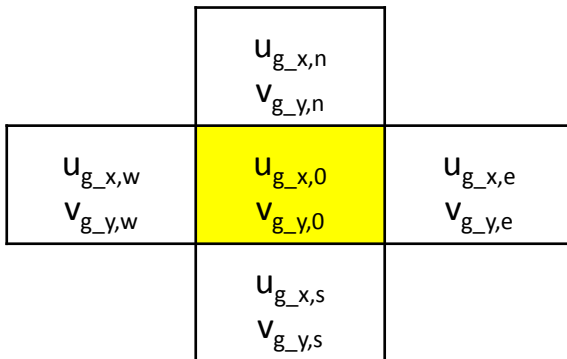


X-direction

$$\begin{aligned}
 & - (P_2 - P_1)_x A_2 - \int_A \tau_{i,x} dA_2 - \int_A \tau_{wg,x} dA_2 \\
 & = \sum_{in\_x} (\rho_g A_2 U_{g\_face} \times U_n) - \sum_{out\_x} (\rho_g A_2 U_{g\_face} \times U_n)
 \end{aligned}$$

Y-direction

$$\begin{aligned}
 & - (P_2 - P_1)_y A_2 + \int_A \tau_{i,y} dA_2 - \int_A \tau_{wg,y} dA_2 + \rho_g g V \\
 & = \sum_{in\_y} (\rho_g A_2 U_{g\_face} \times U_n) - \sum_{out\_y} (\rho_g A_2 U_{g\_face} \times U_n)
 \end{aligned}$$



# Momentum Conservation Equations

X-direction force balance

Liquid film  $-(P_2 - P_1)_x A_1 + \int_A \tau_{i,x} dA_1 - \int_A \tau_{wf,x} dA_1$

$$= \sum_{in\_x} (\rho_f A_1 U_{f\_face} \times U_n) - \sum_{out\_x} (\rho_f A_1 U_{f\_face} \times U_n)$$

Air  $-(P_2 - P_1)_x A_2 - \int_A \tau_{i,x} dA_2 - \int_A \tau_{wg,x} dA_2$

$$= \sum_{in\_x} (\rho_g A_2 U_{g\_face} \times U_n) - \sum_{out\_x} (\rho_g A_2 U_{g\_face} \times U_n)$$

Y-direction force balance

Liquid film  $-(P_2 - P_1)_y A_1 - \int_A \tau_{i,y} dA_1 - \int_A \tau_{wf,y} dA_1 + \rho_f gV$

$$= \sum_{in\_y} (\rho_f A_1 U_{f\_face} \times U_n) - \sum_{out\_y} (\rho_f A_1 U_{f\_face} \times U_n)$$

Air  $-(P_2 - P_1)_y A_2 + \int_A \tau_{i,y} dA_2 - \int_A \tau_{wg,y} dA_2 + \rho_g gV$

$$= \sum_{in\_y} (\rho_g A_2 U_{g\_face} \times U_n) - \sum_{out\_y} (\rho_g A_2 U_{g\_face} \times U_n)$$

# Momentum Conservation Equations

$$\frac{\int_A \tau_{i,x} dA_1}{A_1} - \frac{\int_A \tau_{wf,x} dA_1}{A_1} + \frac{\int_A \tau_{i,x} dA_2}{A_2} + \frac{\int_A \tau_{wg,x} dA_2}{A_2} = \sum_{in_x} (\rho_f U_{f\_face} \times U_n) - \sum_{out_x} (\rho_f U_{f\_face} \times U_n) - \sum_{in_x} (\rho_g U_{g\_face} \times U_n) + \sum_{out_x} (\rho_g U_{g\_face} \times U_n)$$

$$-\frac{\int_A \tau_{i,y} dA_1}{A_1} - \frac{\int_A \tau_{wf,y} dA_1}{A_1} - \frac{\int_A \tau_{i,y} dA_2}{A_2} + \rho_f gH - \rho_g gH = \sum_{in_y} (\rho_f U_{f\_face} \times U_n) - \sum_{out_y} (\rho_f U_{f\_face} \times U_n) - \sum_{in_y} (\rho_g U_{g\_face} \times U_n) + \sum_{out_y} (\rho_g U_{g\_face} \times U_n)$$

$$\int_A \tau_{i,x} dA = \int_A \frac{1}{2} \rho_g f_i \bar{U}_g - \bar{U}_f \left| (U_g - U_f) \right| dA = \int_A \frac{1}{2} \rho_g f_i (u_g(x) - u_f(x)) \sqrt{(u_g(x) - u_f(x))^2 + v_f^2(y)} dA$$

$$\int_A \tau_{wf,x} dA = \int_A \frac{1}{2} \rho_f (f_{wf} u_f(x)) \sqrt{u_f^2(x) + v_f^2(y)} dA$$

$$\int_A \tau_{wg,x} dA = \int_A \frac{1}{2} \rho_g (f_{wg} u_g(x)) dA$$

Blasius' equation

$$f_{wg} = 0.79 \text{Re}^{-1/4}$$

$$\int_A \tau_{i,y} dA = \int_A \frac{1}{2} \rho_g f_i \bar{U}_g - \bar{U}_f \left| (U_g - U_f) \right| dA = \int_A \frac{1}{2} \rho_g f_i v_f(x) \sqrt{(u_g(x) - u_f(x))^2 + v_f^2(y)} dA$$

$$\int_A \tau_{wf,y} dA = \int_A \frac{1}{2} \rho_f (f_{wf} v_f(y)) \sqrt{u_f^2(x) + v_f^2(y)} dA$$

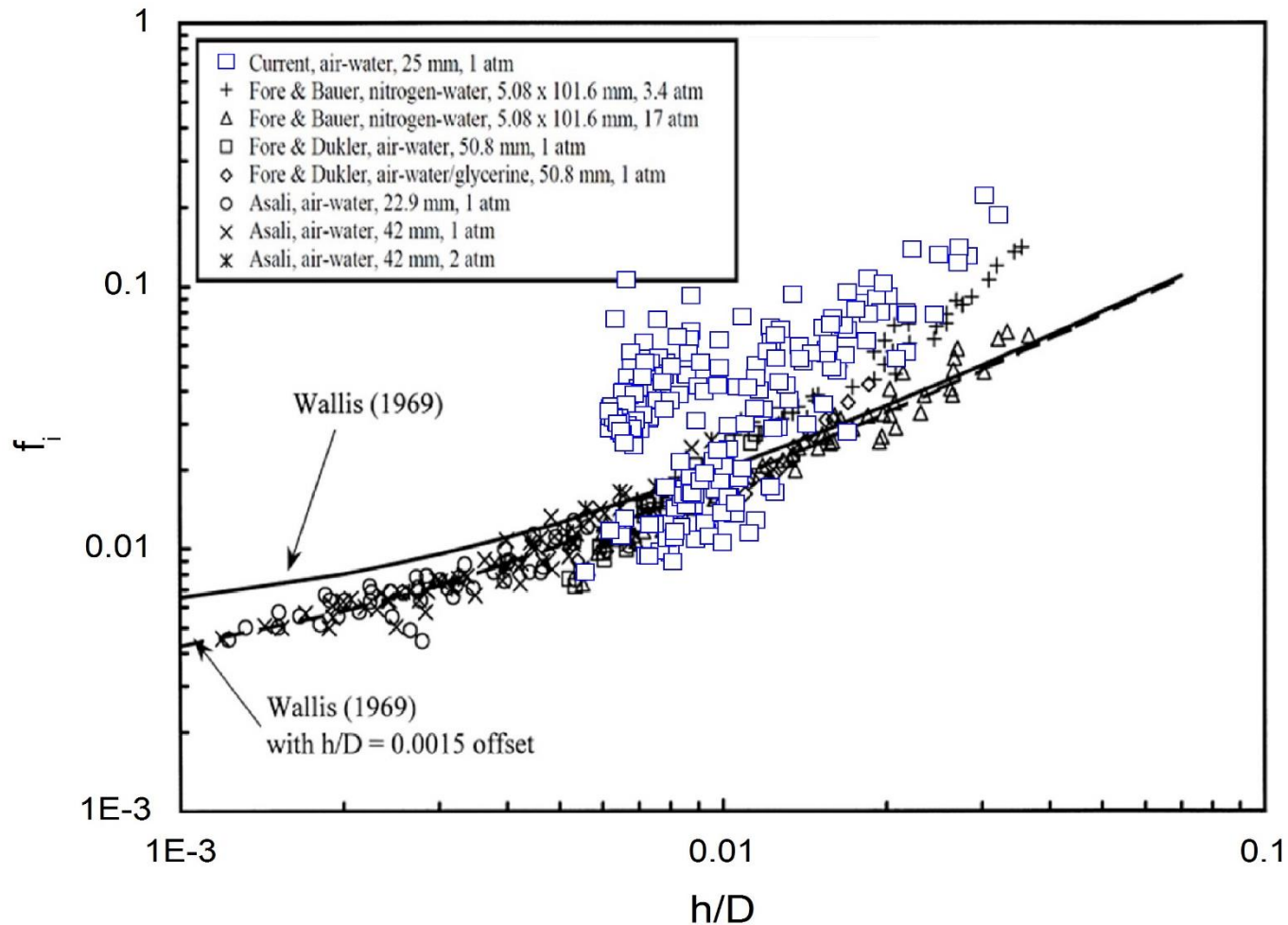
Assume that...

- ① Ignore lateral air flow acceleration
- ② Linear velocity profile
- ③ Constant friction coefficient
- ④ Gas wall friction factor follows Blasius' Eq.



$$\begin{pmatrix} a_1 & a_2 \\ a_3 & a_4 \end{pmatrix} \begin{pmatrix} f_i \\ f_w \end{pmatrix} = \begin{pmatrix} b_1 \\ b_2 \end{pmatrix}$$

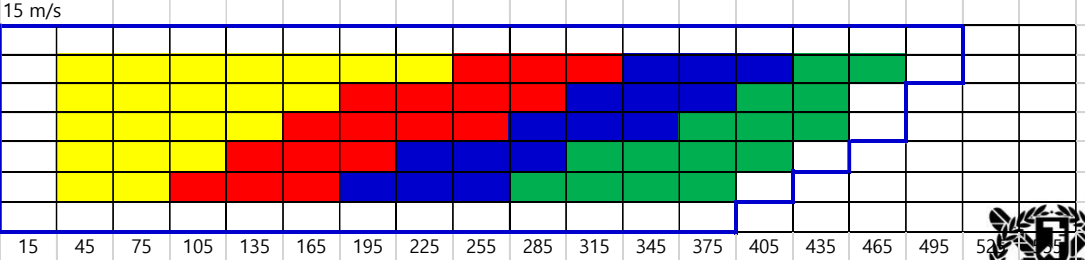
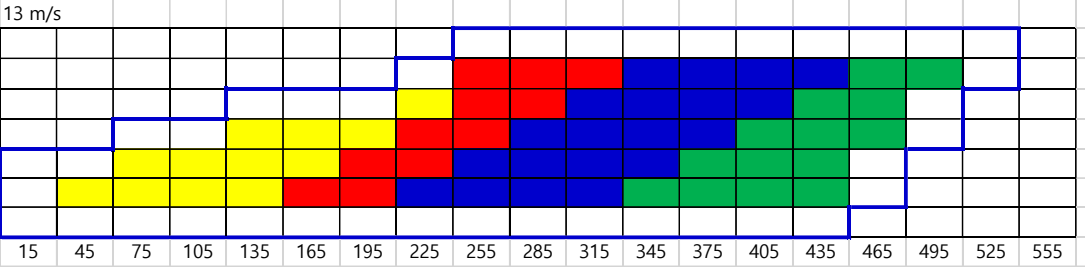
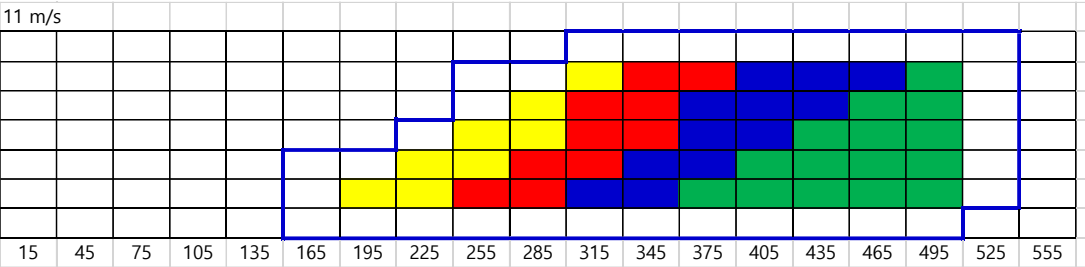
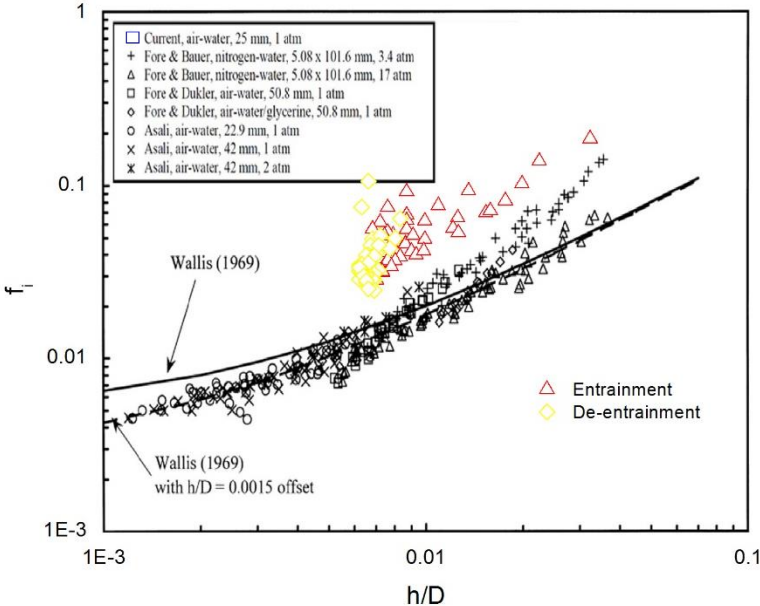
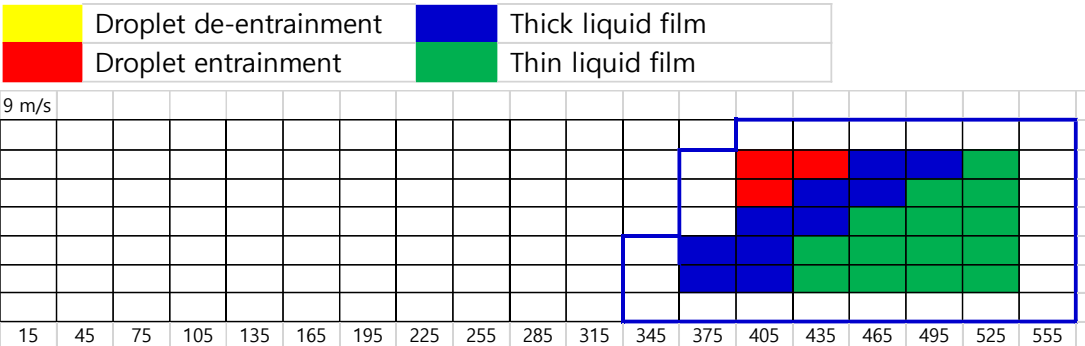
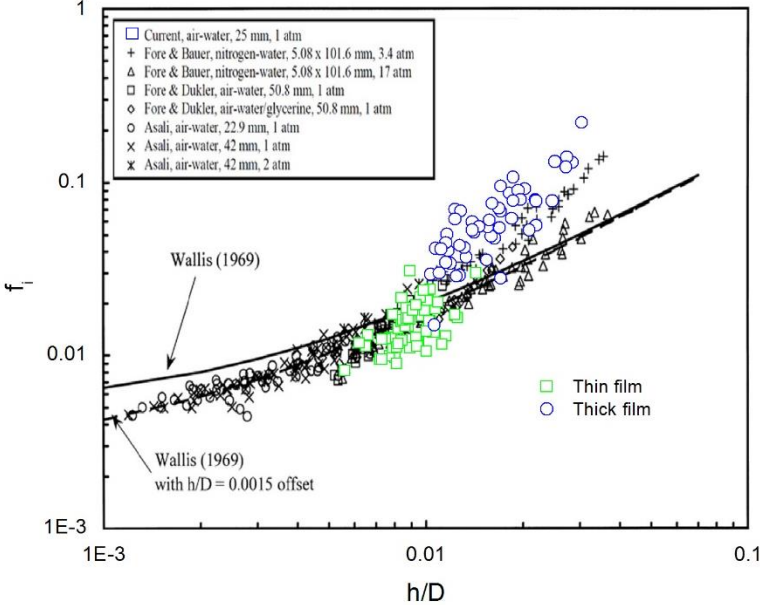
# Local Interfacial Friction Factors vs. Previous Studies



L.B. Fore, S.G. Beus, R.C. Bauer, "Interfacial friction in gas-liquid annular flow: analogies to full and transition roughness", International Journal of Multiphase Flow, Vol. 26, p 1755-1769, 2000.

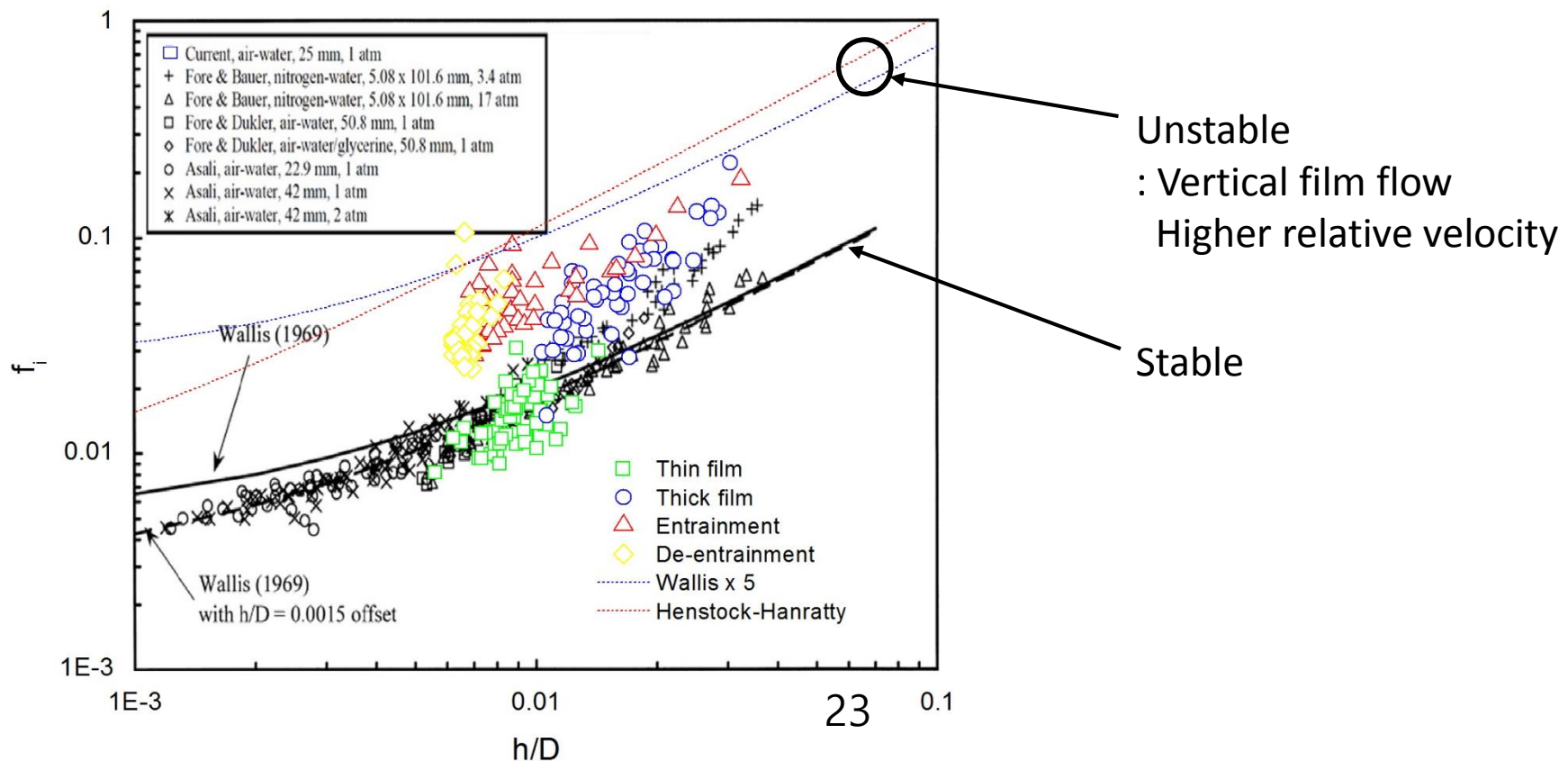


# Flow Pattern Definition



# Interfacial Friction Factor in MARS-3D

Film flow	Stable	$f_i = 0.0025(1 + 75\alpha_1)$	Wallis
	Unstable	$f_i = f_s \left\{ 1 + 1400F \left[ 1 - \exp\left( -\frac{1}{G} \frac{(1 + 1400F)^{3/2}}{13.2F} \right) \right] \right\}$	Use the larger one between <b>Henstock-Hanratty</b> and <b>5×Wallis</b> .



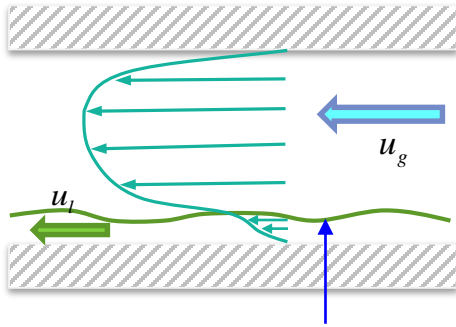
1. Introduction
2. 2D Film Flow Experiments
3. Momentum Conservation Equations

## **4. MECHANISTIC MODEL FOR INTERFACIAL FRICTION FACTOR**

5. Conclusions



# Mechanistic Model for Interfacial Friction Factor



Turbulent boundary layer

$$u^+ = \frac{1}{\kappa} \ln y^+ + B - \Delta B(k_s^+)$$

$$u^+ = \frac{u_g - u_i}{u^*}$$

$B$ : smooth wall single-phase flow constant, 5.5

$\kappa$ : Von Karman constant, 0.4

$\Delta B$ : effects of the rough wall; increases with roughness,  $k_s^+$

$$\Delta B = \frac{1}{\kappa} \ln(1 + Ck_s^+)$$

“Thickness of wavy thin liquid film  $\approx$  wall roughness”

$$\text{Re}_t = \frac{\rho t u_{\text{avg}}}{\mu} = \int_0^{t^+} u^+ dy^+ = \frac{t^+}{\kappa} (\ln t^+ + B\kappa - \Delta B\kappa - 1)$$

$$f_i = 2 \frac{u^{*2}}{u_{\text{avg}}^2} = 2 \left( \frac{\rho t u^*}{\mu} \right)^2 \left( \frac{\mu}{\rho t u_{\text{avg}}} \right)^2 = 2 \frac{t^{+2}}{\text{Re}_t^2}$$

where  $u^* = \sqrt{\tau_i / \rho_g}$

$$\Rightarrow \text{Re}_t = \sqrt{\frac{2}{f_i}} t^+$$

$$\therefore \sqrt{\frac{2}{f_i}} = \frac{1}{\kappa} \left[ \ln \left( \frac{t^+}{1 + Ck_s^+} \right) + B\kappa - 1 \right]$$

R.Kumar and D.P. Edwards, “Interfacial shear modeling in two-phase annular flow”, KAPL Atomic Power Laboratory, KAPL-P-000220, 1996.

# Mechanistic Model for Interfacial Friction Factor

## ❖ New approach with local experimental results

$$\therefore \sqrt{\frac{2}{f_i}} = \frac{1}{\kappa} \left[ \ln \left( \frac{t^+}{1 + Ck_s^+} \right) + B\kappa - 1 \right]$$



$$0.4 \times \sqrt{\frac{2}{f_i}} = \ln \left( \frac{\text{Re}_t \sqrt{\frac{f_i}{2}}}{1 + C \text{Re}_t \sqrt{\frac{f_i}{2}} \frac{k_s}{t}} \right) + 1.2$$



$$f_i = \frac{1}{4} \left[ 1.879 - 4.912C \frac{k_s}{t} \right]^{-2}$$

$$D = 2t, \quad k_s \approx 4h$$

$$t^+ = \text{Re}_t \sqrt{\frac{f_i}{2}}$$

$$k_s^+ = \frac{k_s u^*}{\nu} = \frac{tu^*}{\nu} \frac{k_s}{t} = \frac{tu_{avg}}{\nu} \frac{u^*}{u_{avg}} \frac{k_s}{t} = \text{Re}_t \sqrt{\frac{f_i}{2}} \frac{k_s}{t}$$

Assume that...

- ① Modeling constant(C) = 6.5(unstable) / 4.5(stable)
- ② For large  $\text{Re}_t \sqrt{4f_i}$
- ③  $k_s/t \ll 1$

$$f_i = \frac{1}{4} \left[ 1.879 - 22.104 \left( \frac{h}{D} - 0.0015 \right) \right]^{-2} - 0.066$$

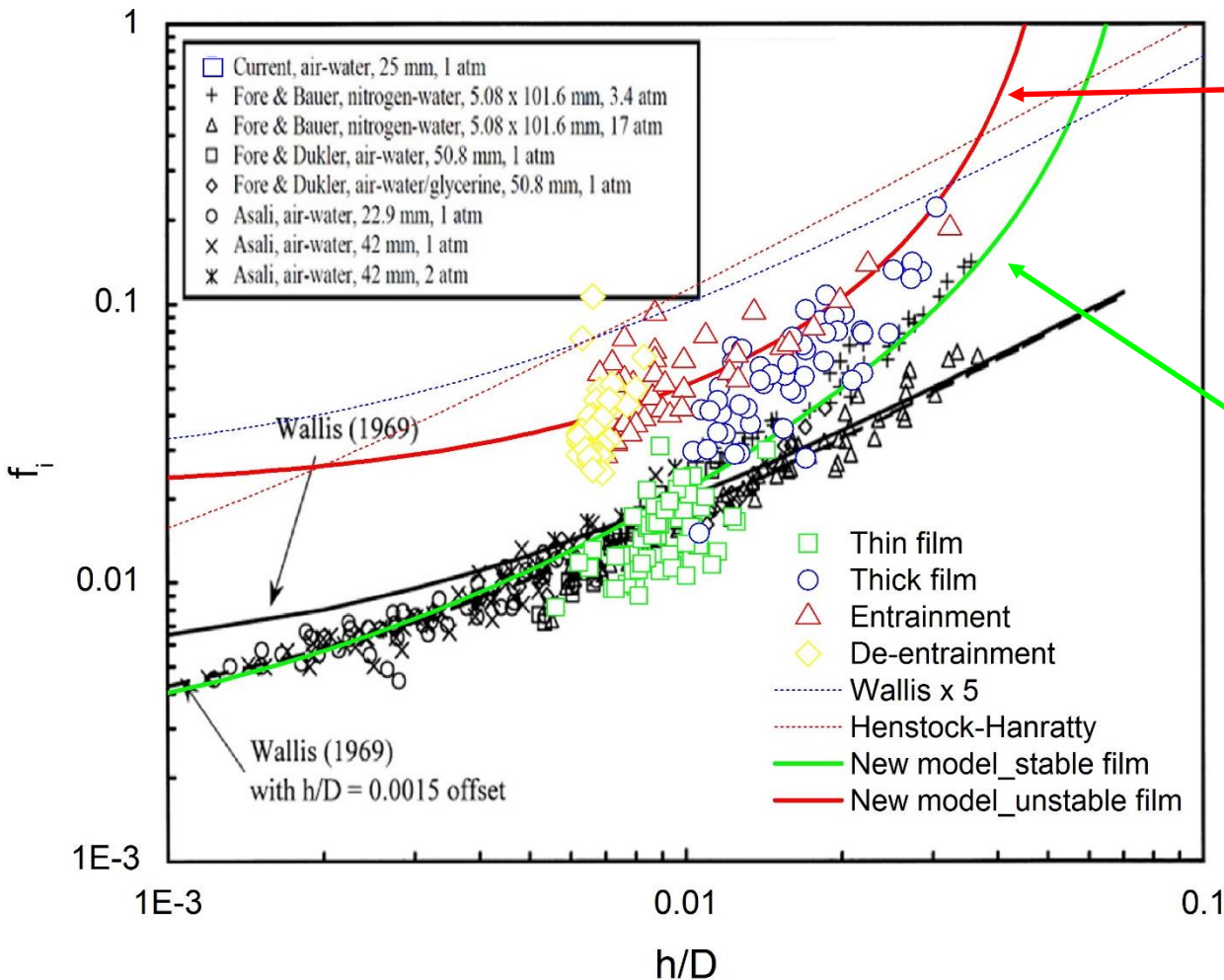
Initial point for single gas flow

( $f_{sp} = 0.005$  in stable film)

$$f_i = \frac{1}{4} \left[ 1.879 - 32.928 \left( \frac{h}{D} - 0.0015 \right) \right]^{-2} - 0.046$$

( $5 \times f_{sp} = 0.025$  in unstable film)

# Comparison with Previous Studies



$$f_i = \frac{1}{4} \left[ 1.879 - 32.928 \left( \frac{h}{D} - 0.0015 \right) \right]^{-2} - 0.046$$

Unstable film flow  
(entrainment, de-entrainment)

$$f_i = \frac{1}{4} \left[ 1.879 - 22.104 \left( \frac{h}{D} - 0.0015 \right) \right]^{-2} - 0.066$$

Stable film flow  
(thin film, thick film w/o entrainment)

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4. Mechanistic Model for Interfacial Friction Factor

## 5. CONCLUSIONS

## ❖ Summary

- Local 2D liquid film velocity & thickness measurement were carried out.  
→ Validation data for multidimensional codes were produced.
- Interfacial friction factors were calculated by 2D momentum conservation equations.
- Mechanistic interfacial friction factor model was developed based on law of wall and thin film roughness concept.

Stable film flow (thin film, thick film w/o entrainment)	$f_i = \frac{1}{4} \left[ 1.879 - 22.104 \left( \frac{h}{D} - 0.0015 \right) \right]^{-2} - 0.066$
Unstable film flow (entrainment, de-entrainment)	$f_i = \frac{1}{4} \left[ 1.879 - 32.928 \left( \frac{h}{D} - 0.0015 \right) \right]^{-2} - 0.046$

## ❖ Further work

- ✓ Film thickness measurement error in entrainment & de-entrainment region
- ✓ Lateral air velocity acceleration
- ✓ Gas wall friction
- ✓ Transition criterion between stable/unstable film

# APPENDIX

1. Experimental methods
2. Mass and Momentum Conservation
3. Uncertainty Analysis
4. Challenges

# EXPERIMENTAL METHODS

# Application of Volume-PIV Method

## ❖ Volume-averaged PIV measurement method

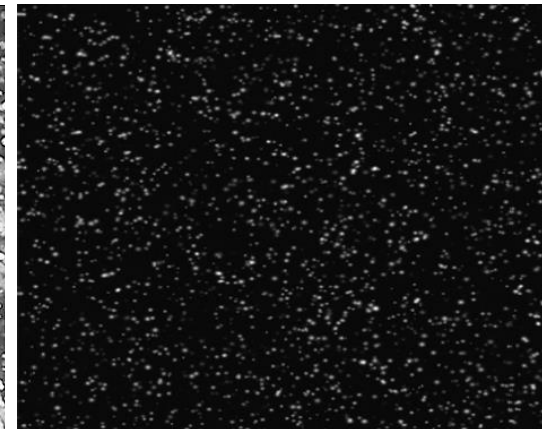
- Continuous wave lasers: 532 nm laser(green) / 10W + 5W
- High speed camera (PHANTOM v211)
- Add fluorescent particles (1~20  $\mu\text{m}$ ).
  - : luminous particles in range of 570 nm wavelength reflecting 532 nm laser source.
- Use a long-pass filter which cuts off image information under 560 nm wavelength.



Set-up of volume-PIV method



Original image



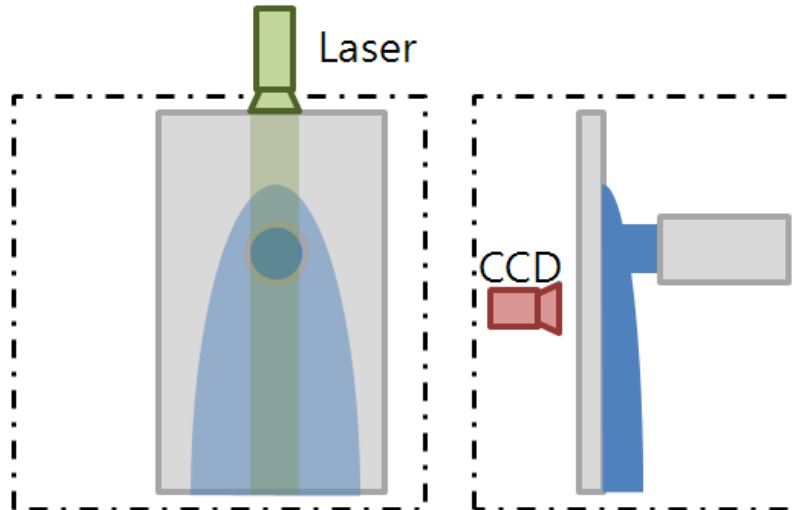
Filtered image



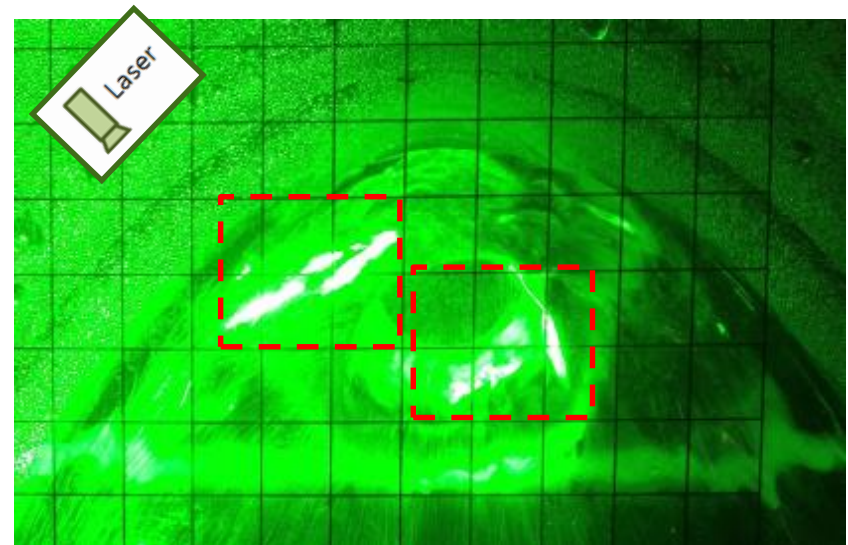
# Limitation of Sheet-PIV Method

## ❖ Why the sheet-PIV is not appropriate method to measure liquid film velocity...

- Geometry
  - Laser sheet cannot help being injected along to the vertical direction.
- Optics
  - Laser is scattered by the oscillating film boundary.
  - Attenuation of laser intensity doesn't maintain illumination.



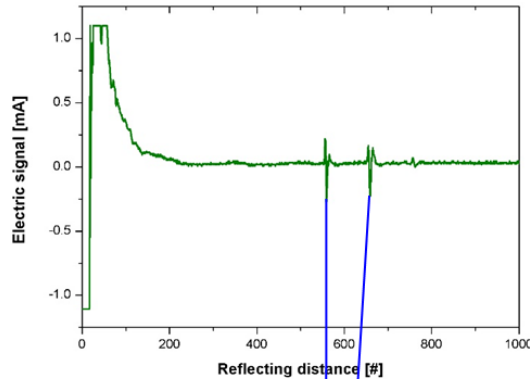
Application of sheet-PIV on liquid film flow



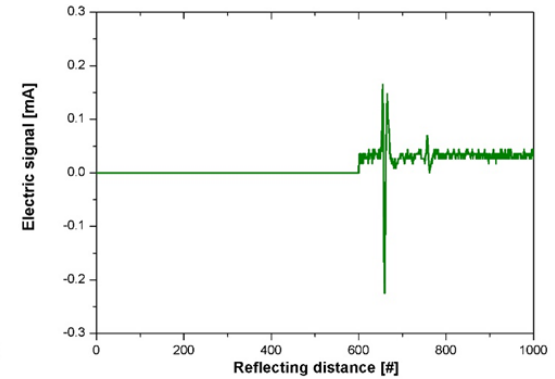
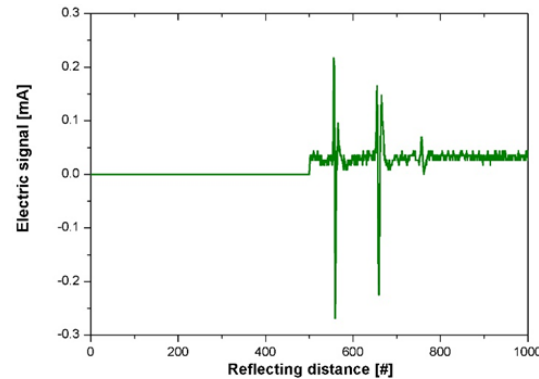
# Liquid Film Thickness Measurement

## ❖ Ultrasonic velocity gauge

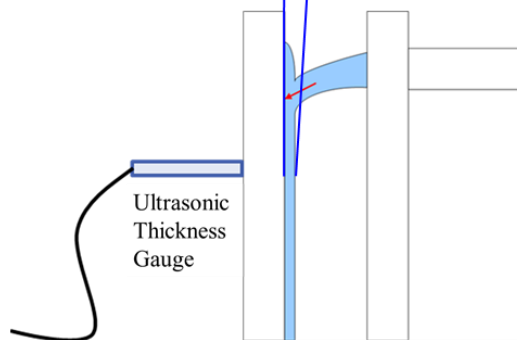
- Signal frequency: 50MHz / Sample rate: 20Hz (200samples per 10seconds)



Raw data



- ① Remove acryl
- ② Rescaling
- ③ Find first peak point
- ④ Remove tails
- ⑤ Find second peak point



$$\delta = \frac{v_u}{2} \left( \frac{x_{2p} - x_{1p}}{f_{signal}} \right)$$

$v_u$  : Ultrasonic velocity of water

$x_{2p} - x_{1p}$  : gap distance

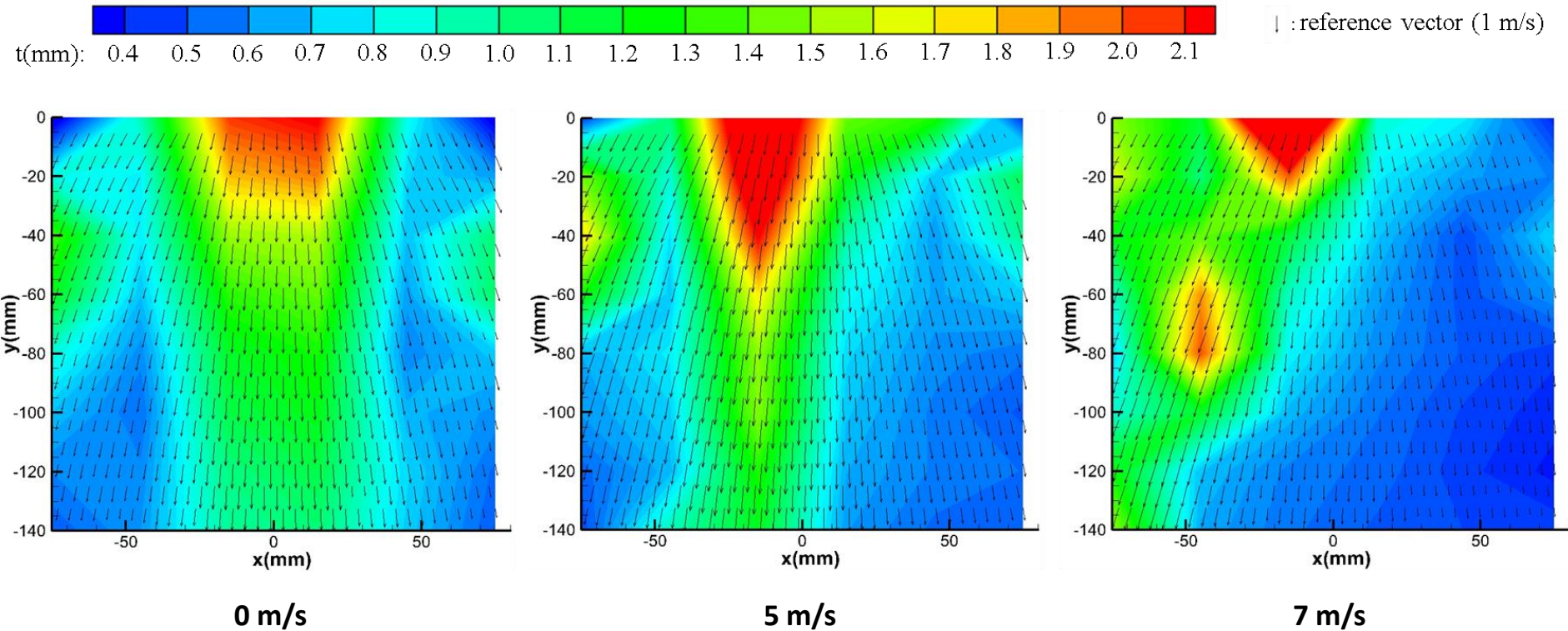
$f_{signal}$  : Signal frequency (50MHz)

$$v_u = 1402.38744 + 5.03836171 \times T_w - 0.0581172916 \times T_w^2 + 0.000334638117 \times T_w^3 - 0.00000148259672 \times T_w^4 + 0.0000000031658502 \times T_w^5$$

# MASS CONSERVATION

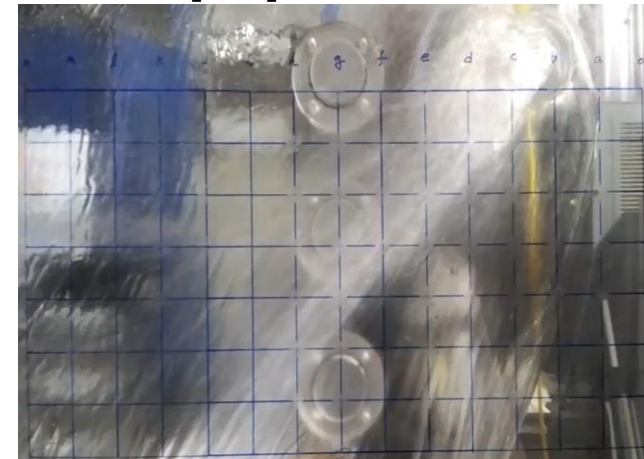
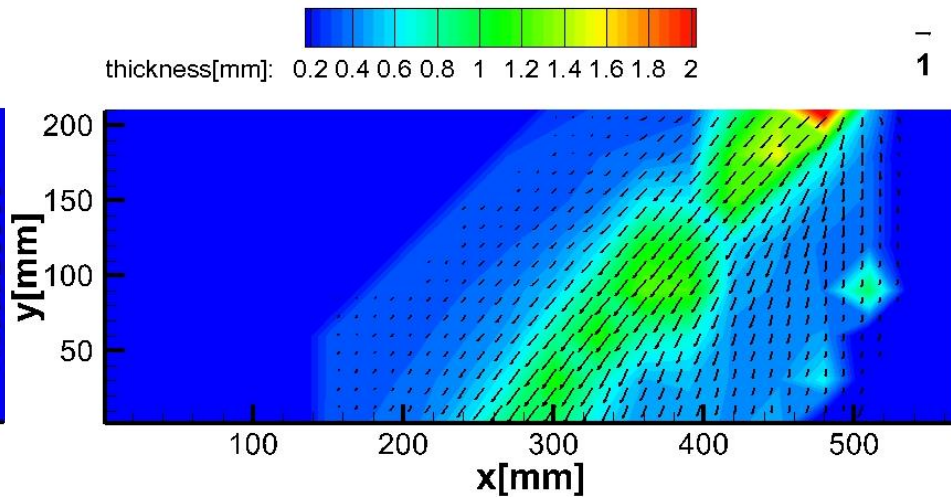
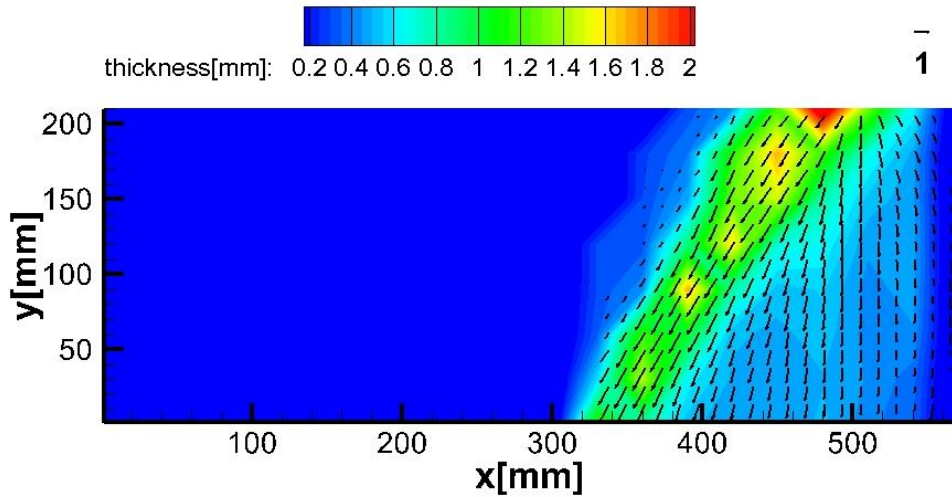
# 2D Film Flow Experiments

## Results: 0, 5, 7 m/s lateral air injection



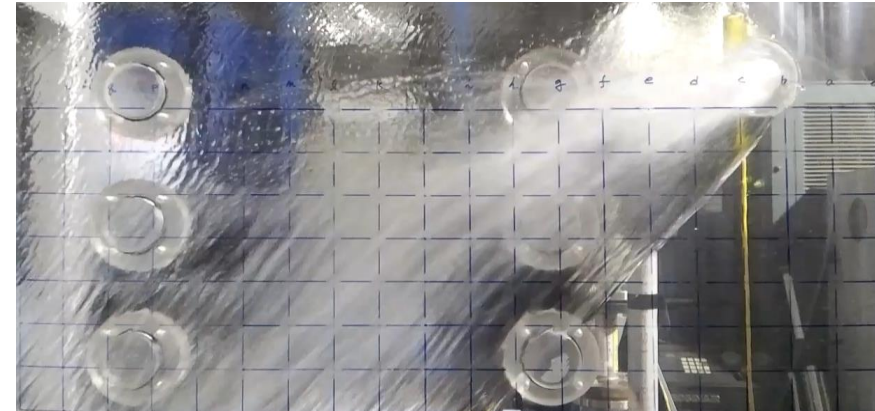
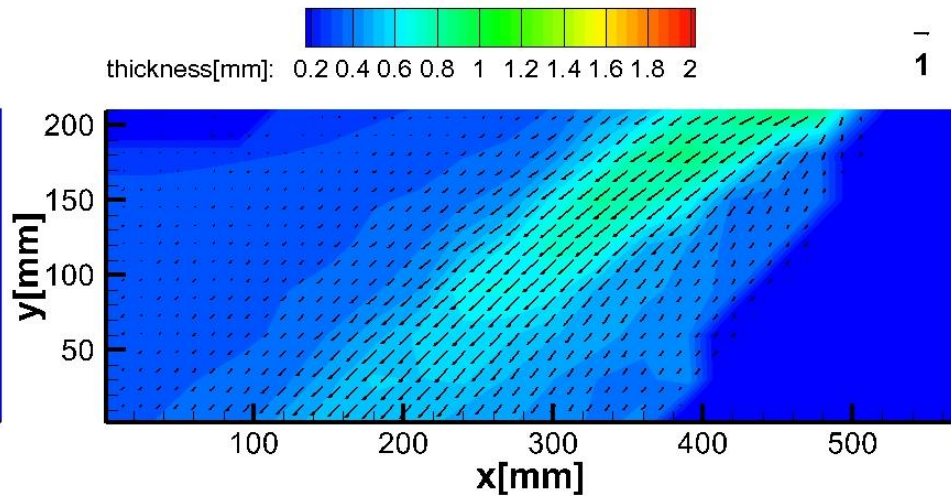
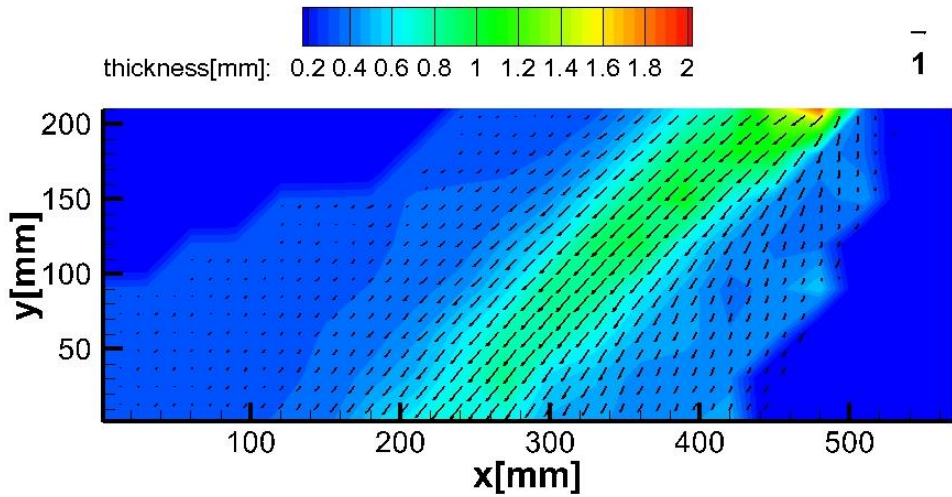
# Results of 2D Film Flow Experiments

## ❖ Results: 9, 11, 13, 15 m/s lateral air injection (1/2)



# Results of 2D Film Flow Experiments

## Results: 9, 11, 13, 15 m/s lateral air injection (2/2)

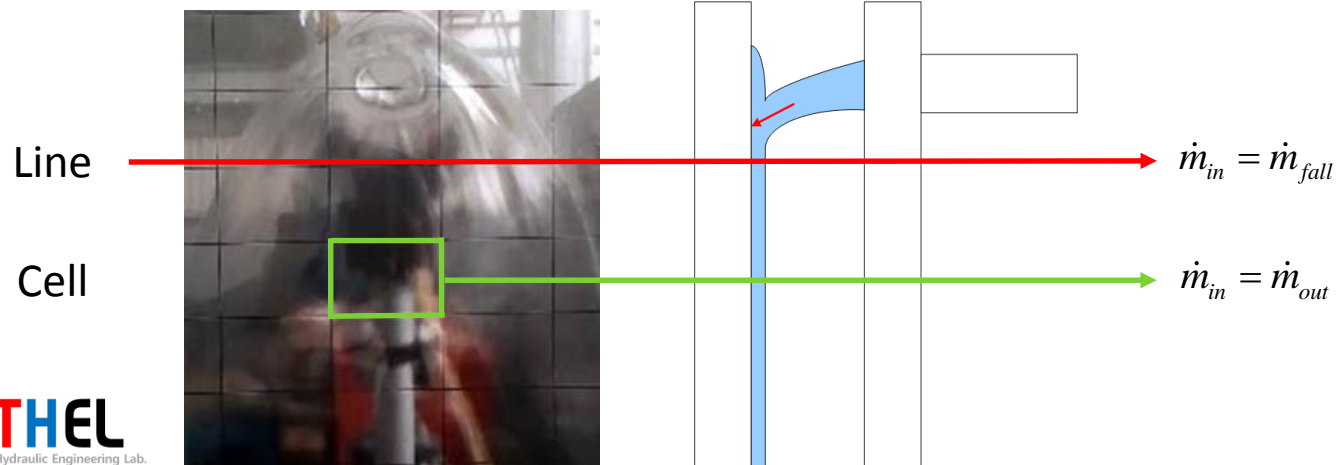


13 m/s

15 m/s

# Mass Conservation

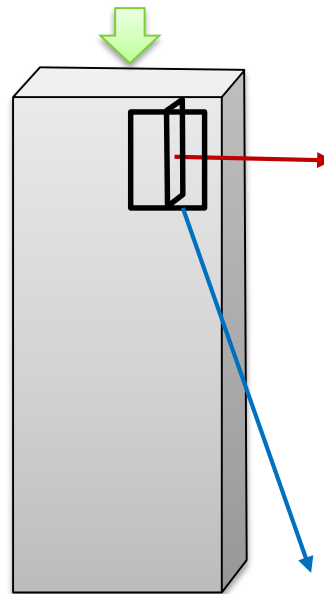
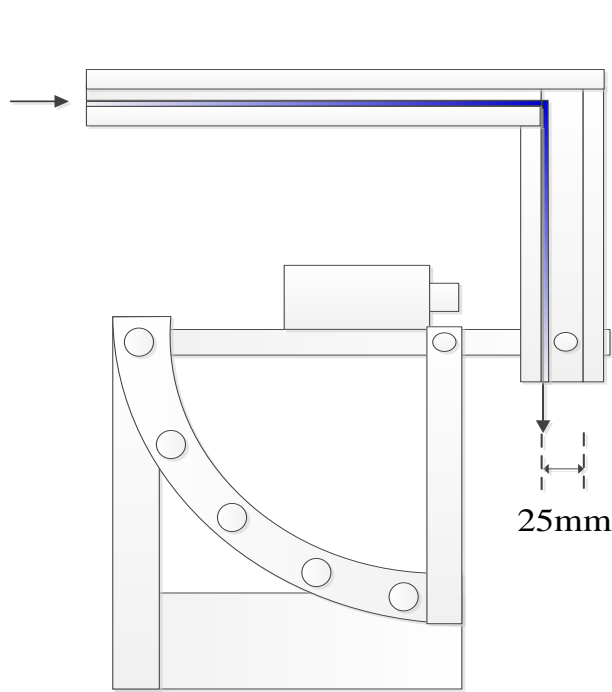
Mass conservation[error=(M_in-M_out)/M_in]		0	5	7	9	11	13	15
Lateral air velocity								
Line	M_out	0.00024	0.00025	0.00021	0.00014	0.00011	0.00011	0.00009
	error	0.0017	0.0522	0.1370	0.3985	0.5305	0.5502	0.6212
	%	0.17	5.22	13.70	39.85	53.05	55.02	62.12
					middle			
					0.00018			
					0.2482			
					24.82			
					lower			
					0.00018			
					0.2441			
					24.41			
Cell	Max.	0.122	0.122	0.198	0.195	0.244	0.274	0.216
	Min.	0.002	0.001	0.002	0.003	0.002	0.000	0.002
	Avg.	0.040	0.048	0.049	0.084	0.091	0.070	0.060
	Stdv.	0.032	0.036	0.053	0.049	0.067	0.056	0.050



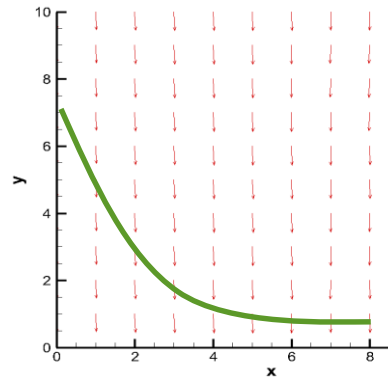
# UNCERTAINTY ANALYSIS



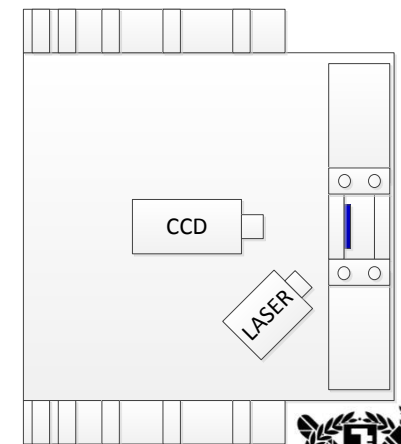
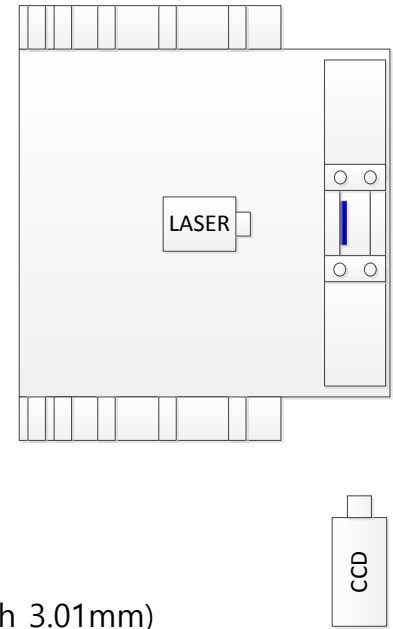
## ❖ Comparison between sheet-PIV and volume-PIV



Test section

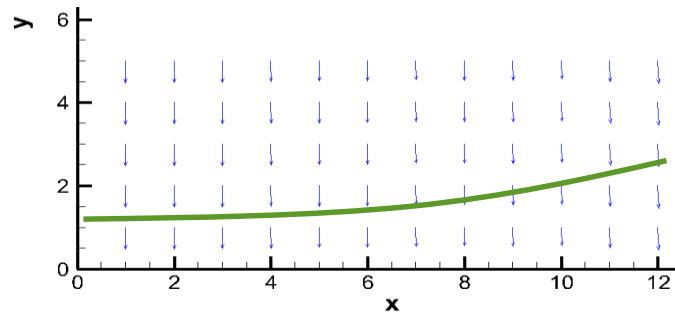


Side view with sheet PIV(depth 3.01mm)



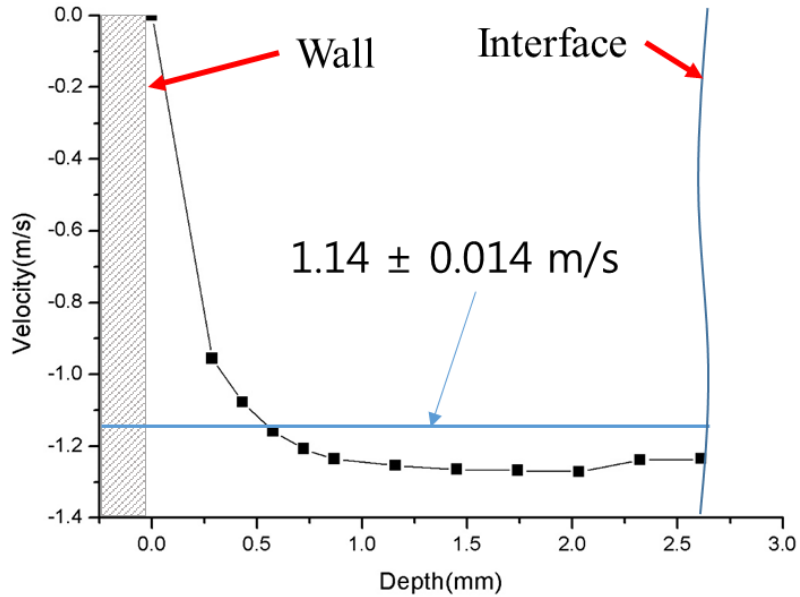
### Schematic of validation experimental facility

- ✓ Liquid film flow (~ 3mm)
- ✓ Sheet-PIV vs. volume PIV
- ✓ Uncertainty quantification

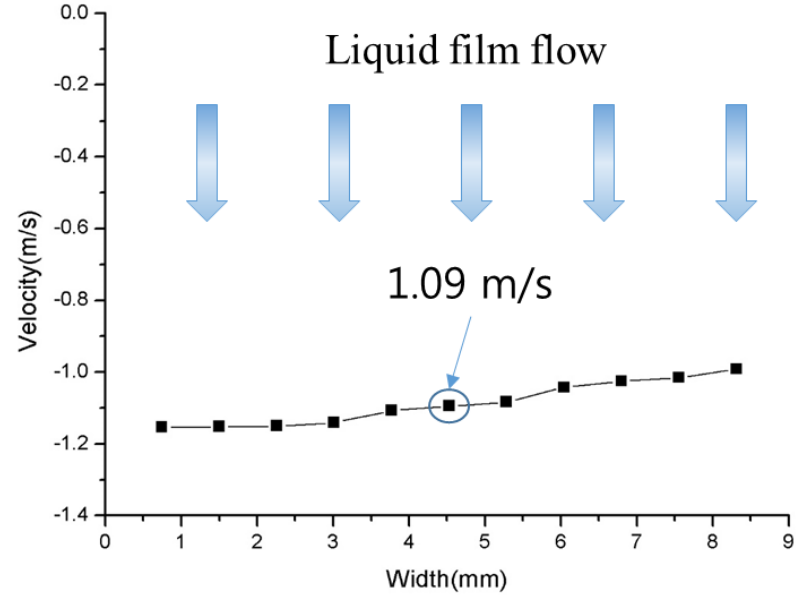


Front view with volume PIV(width 11.85mm)

## Results



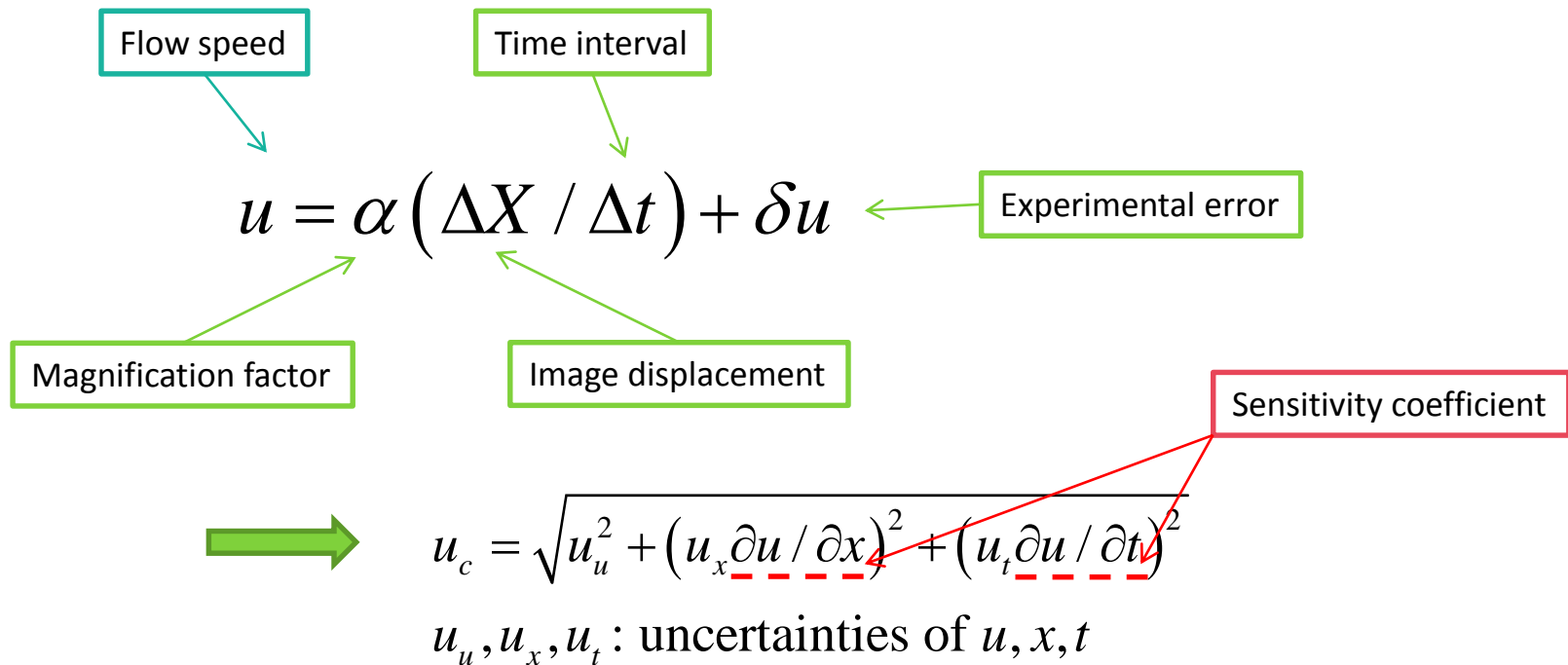
Sheet-PIV along to depth direction



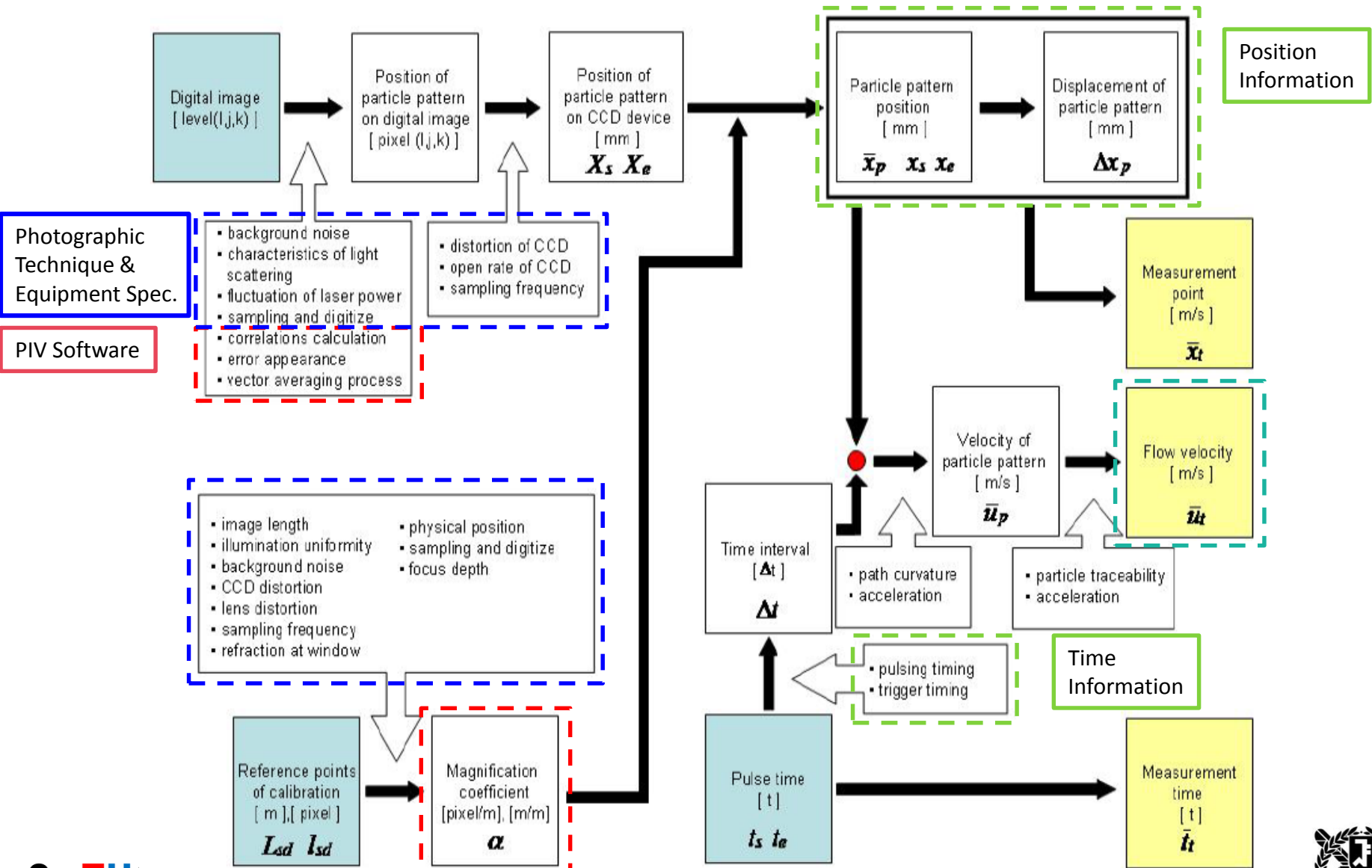
Volume-PIV along to width direction

## ❖ Uncertainty Analysis of Particle Image Velocimetry

- Recommended UA procedure was developed by Nishio et al. at 1999.
- Uncertainty of flow speed can be defined by image displacement, time interval and magnification factor.
- Experimental errors should be considered as uncertainty parameters.



# General Guidelines



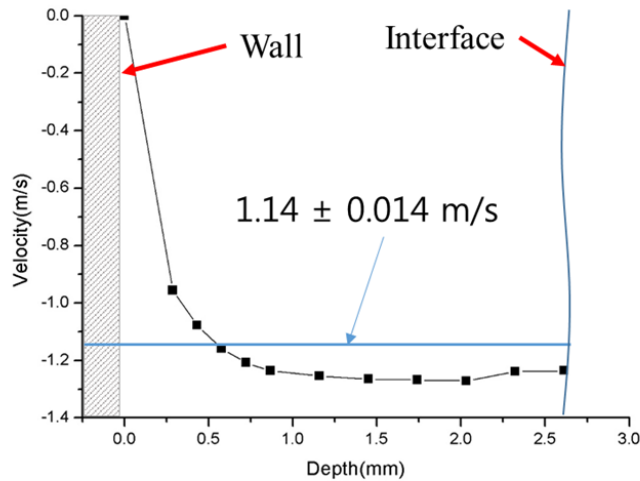
# Summary of Uncertainties

parameter	Category	Error sources	$u(x_i)$	unit	$c_i(\text{unit})$	unit	$c_i u(x_i)$	$u_c$
$\alpha(\text{mm/pix})$	Calibration	Reference image	0.7	pix	1.29E-05	mm/pix <sup>2</sup>	9.01E-06	5.37E-05
		Physical distance	0.02	mm	1.60E-03	1/pix	3.21E-05	
		Image distortion by lens	3.12	pix	1.29E-05	mm/pix <sup>2</sup>	4.01E-05	
		Image distortion by CCD	0.0056	pix	1.29E-05	mm/pix <sup>2</sup>	7.21E-08	
		Board position	0.5	mm	1.51E-05	1/pix	7.55E-06	
		Parallel board	0.035	rad	2.81E-04	mm/pix	9.84E-06	
$\Delta X(\text{pix})$	Acquisition	Particle image distortion	0.000002	mm	1.25E+02	pix/mm	2.49E-04	0.20
		Image distortion by CCD	0.0056	pix	1.29E-05		7.21E-08	
		Normal view angle	0.035	rad	2.81E-04	mm/pix	9.84E-06	
$\Delta t(\text{s})$	Reduction	Mis-matching error	0.2	pix	1.00E+00		0.20	0.20
		Sub-pixel analysis	0.03	pix	1.00E+00		0.03	
$\Delta t(\text{s})$	Acquisition	Pulse timing accuracy	4.00E-08	s	1.00E+00		4.00E-08	4.00E-08
$\delta u(\text{mm/s})$	Experiment	Particle trajectory	0.1	mm/s	1.00E+00		0.1	0.10
		3-D effects	1.10E-03	mm/s	1.00E+00		0.0011	

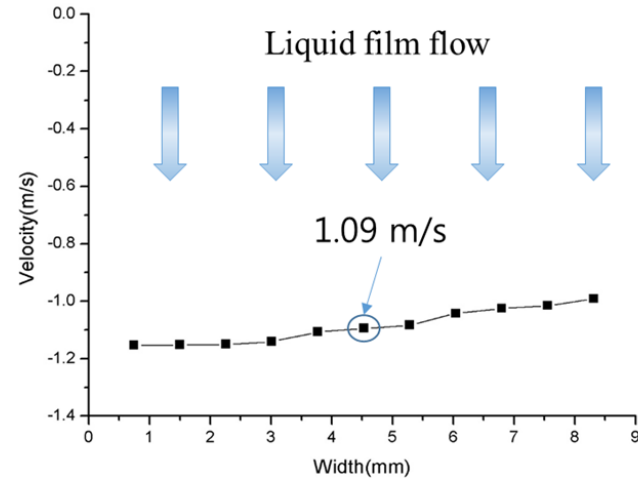
parameter	Error sources	$u(x_i)$	unit	$c_i(\text{unit})$	unit	$c_i u(x_i)$	$u_c$
$\alpha$	Magnification factor	5.363E-05	mm/pix	158461.1	pix/s	8.4982082	
$\Delta X$	Image displacement	0.20223764	pix	17.644	mm/pix/s	3.5682809	
$\Delta t$	Image interval	4.005E-08	s	2795887.6	mm/s <sup>2</sup>	0.1119752	
$\delta u$	Experiment	0.10000605	mm/s	1		0.100006	9.22

- ✓ Averaged velocity: 1.14 m/s
- ✓ Systematic error:  $9.22 \times 10^{-3}$  m/s (0.81%)

## ❖ Uncertainty quantification



Sheet-PIV along to depth direction



Volume-PIV along to width direction

- Averaged standard deviation

$$2\bar{\sigma}_1 = \frac{\sigma_{RSS}}{n} = 0.014 \text{ m/s (1.23\%)}$$

- Total error of volume PIV can be defined as 5.65%.

Averaged velocity = **1.09 ± 0.06 m/s**

$$B = (\bar{V}_1 + 2\bar{\sigma}_1) - \bar{V}_2 = 0.063 \text{ m/s (5.53\%)}$$

where  $\bar{V}_1$  : average velocity by sheet PIV

$\bar{V}_2$  : average velocity by volume PIV

$\sigma_1$  : averaged standard deviation of sheet PIV

# CHALLENGES

## 1. Geometry: Cylindrical vs. parallel plates

- 1/10 scale down can bring hydraulic distortion about interaction between two-phase flow.
- Experimental technique(volume-averaged PIV) has limitation to be applied to curved geometry.

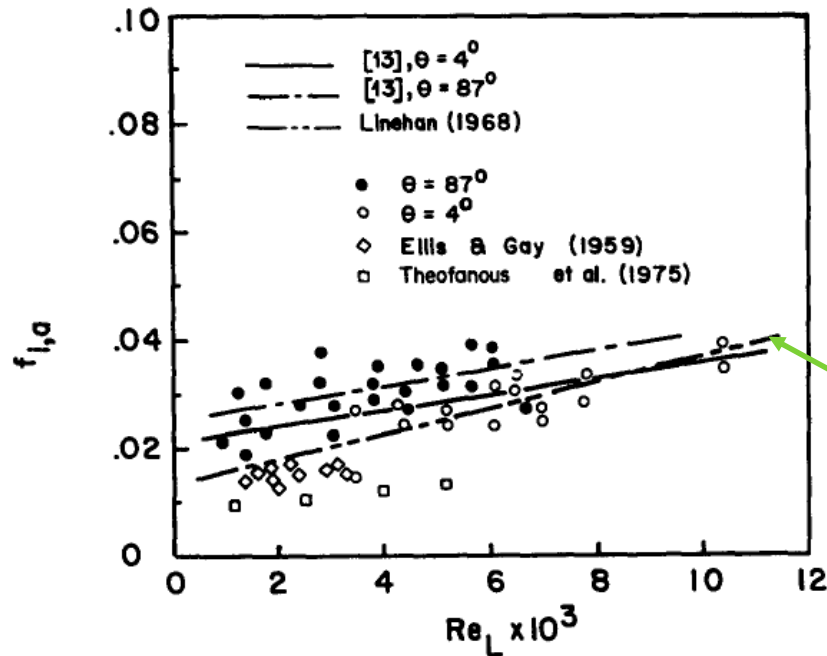
## 2. Scaling: 1/10 scaled down experimental condition

- 1/5 scale experiments will be carried out.
- Other results of previous 1/5 scaled experiments (MIDAS and DIVA) will be validated.



## 3. Fluid: Steam vs. air

- Steam blocks our view on liquid flow.
- Previous studies on steam & saturated water flow shows similar range of interfacial friction factor.



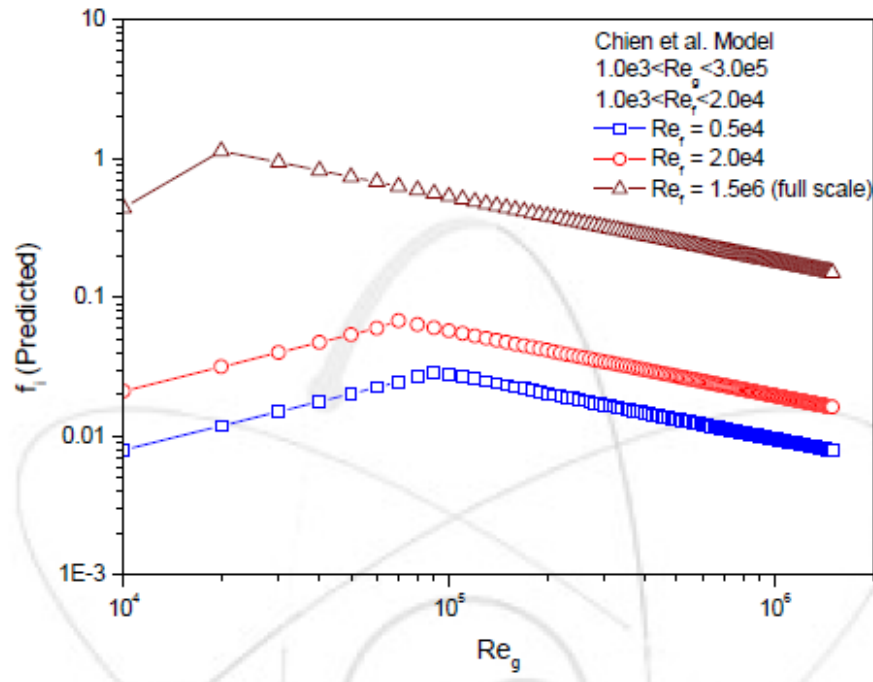
Air-water flow

Steam-water interfacial friction factor

H. J. KIM, Chang, S. C. LEE, S. G. BANKOFF, 1985. "HEAT TRANSFER AND INTERFACIAL DRAG IN COUNTERCURRENT STEAM-WATER STRATIFIED FLOW". International Journal of Multiphase Flow, Vol. 11, pp. 593-606.

## 4. Other modeling parameter (1/2)

- Re number, pressure, etc...
- Interfacial friction factor increase with Re number increases.

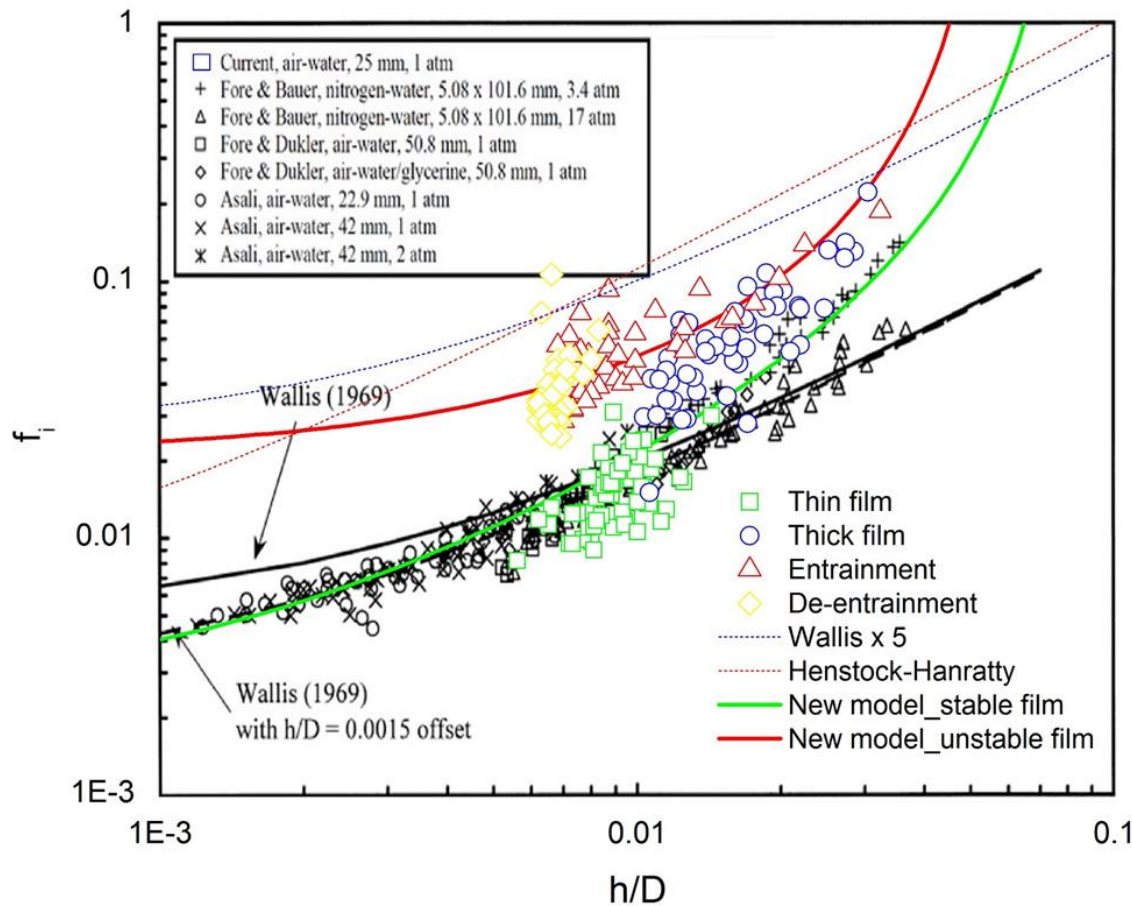


Interfacial friction factor with Re

조형규, 윤병조, 어동진, 권태순, 2005. "LBLOCA 재관수 기간 동안 DVI 방식 원자로에서 발생하는 다차원 안전주입수 우회 현상에 대한 실험 및 이론 연구". KAERI/TR-2937/2005.

## 4. Other modeling parameter (2/2)

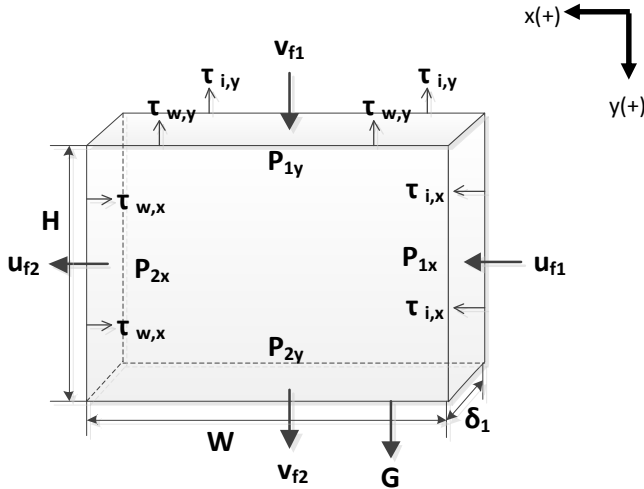
- Re number, pressure, hydraulic diameter etc...
- Results of high pressure experiments has good agreement with Wallis model.



Pressure effect ??

# Momentum Conservation Equations

## Force balance of liquid film

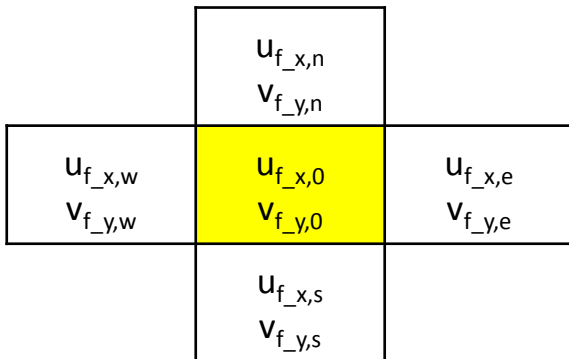


X-direction

$$\begin{aligned}
 & -(P_2 - P_1)_x \delta_1 H + \int_A \tau_{i,x} dA - \int_A \tau_{wf,x} dA \\
 & = -\rho_f \left( \frac{u_{f-e} + u_{f-0}}{2} \right) \delta_1 H \times \left( \frac{u_{f-e} + u_{f-0}}{2} \right) + \rho_f \left( \frac{u_{f-0} + u_{f-w}}{2} \right) \delta_1 H \times \left( \frac{u_{f-0} + u_{f-w}}{2} \right) \\
 & - \rho_f \left( \frac{u_{f-n} + u_{f-0}}{2} \right) \delta_1 W \times \left( \frac{v_{f-n} + v_{f-0}}{2} \right) + \rho_f \left( \frac{u_{f-0} + u_{f-s}}{2} \right) \delta_1 W \times \left( \frac{v_{f-0} + v_{f-s}}{2} \right)
 \end{aligned}$$

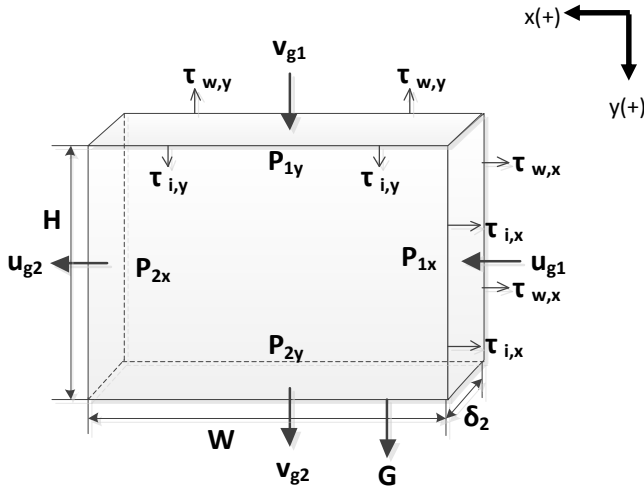
Y-direction

$$\begin{aligned}
 & -(P_2 - P_1)_y \delta_1 W - \int_A \tau_{i,y} dA - \int_A \tau_{wf,y} dA + \rho_f g \delta_1 H W \\
 & = -\rho_f \left( \frac{v_{f-n} + v_{f-0}}{2} \right) \delta_1 W \times \left( \frac{v_{f-n} + v_{f-0}}{2} \right) + \rho_f \left( \frac{v_{f-0} + v_{f-s}}{2} \right) \delta_1 W \times \left( \frac{v_{f-0} + v_{f-s}}{2} \right) \\
 & - \rho_f \left( \frac{v_{f-e} + v_{f-0}}{2} \right) \delta_1 H \times \left( \frac{u_{f-e} + u_{f-0}}{2} \right) + \rho_f \left( \frac{v_{f-0} + v_{f-w}}{2} \right) \delta_1 H \times \left( \frac{u_{f-0} + u_{f-w}}{2} \right)
 \end{aligned}$$



# Momentum Conservation Equations

## Force balance of air

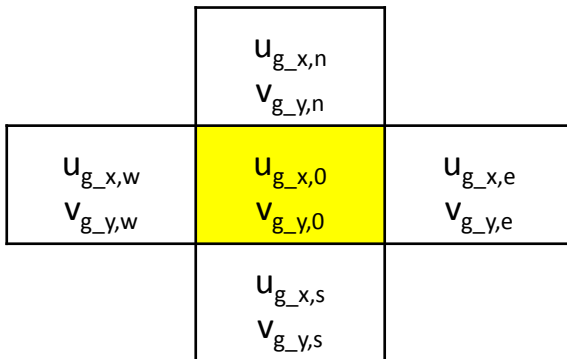


X-direction

$$\begin{aligned}
 & -(P_2 - P_1)_x \delta_2 H - \int_A \tau_{i,x} dA - \int_A \tau_{wg,x} dA \\
 & = -\rho_g \left( \frac{u_{g-e} + u_{g-0}}{2} \right) \delta_2 H \times \left( \frac{u_{g-e} + u_{g-0}}{2} \right) + \rho_g \left( \frac{u_{g-0} + u_{g-w}}{2} \right) \delta_2 H \times \left( \frac{u_{g-0} + u_{g-w}}{2} \right) \\
 & - \rho_g \left( \frac{u_{g-n} + u_{g-0}}{2} \right) \delta_2 W \times \left( \frac{v_{g-n} + v_{g-0}}{2} \right) + \rho_g \left( \frac{u_{g-0} + u_{g-s}}{2} \right) \delta_2 W \times \left( \frac{v_{g-0} + v_{g-s}}{2} \right)
 \end{aligned}$$

Y-direction

$$\begin{aligned}
 & -(P_2 - P_1)_y \delta_2 W + \int_A \tau_{i,y} dA - \int_A \tau_{wg,y} dA + \rho_g g \delta_2 H W \\
 & = -\rho_g \left( \frac{v_{g-n} + v_{g-0}}{2} \right) \delta_2 W \times \left( \frac{v_{g-n} + v_{g-0}}{2} \right) + \rho_g \left( \frac{v_{g-0} + v_{g-s}}{2} \right) \delta_2 W \times \left( \frac{v_{g-0} + v_{g-s}}{2} \right) \\
 & - \rho_g \left( \frac{v_{g-e} + v_{g-0}}{2} \right) \delta_2 H \times \left( \frac{u_{g-e} + u_{g-0}}{2} \right) + \rho_g \left( \frac{v_{g-0} + v_{g-w}}{2} \right) \delta_2 H \times \left( \frac{u_{g-0} + u_{g-w}}{2} \right)
 \end{aligned}$$



# Momentum Conservation Equations

## X-direction force balance

Liquid film  $-(P_2 - P_1)_x \delta_1 H + \int_A \tau_{i,x} dA - \int_A \tau_{wf,x} dA = -\rho_f \left( \frac{u_{f-e} + u_{f-0}}{2} \right) \delta_1 H \times \left( \frac{u_{f-e} + u_{f-0}}{2} \right) + \rho_f \left( \frac{u_{f-0} + u_{f-w}}{2} \right) \delta_1 H \times \left( \frac{u_{f-0} + u_{f-w}}{2} \right) - \rho_f \left( \frac{u_{f-n} + u_{f-0}}{2} \right) \delta_1 W \times \left( \frac{v_{f-n} + v_{f-0}}{2} \right) + \rho_f \left( \frac{u_{f-0} + u_{f-s}}{2} \right) \delta_1 W \times \left( \frac{v_{f-0} + v_{f-s}}{2} \right)$

Air  $-(P_2 - P_1)_x \delta_2 H - \int_A \tau_{i,x} dA - \int_A \tau_{wg,x} dA = -\rho_g \left( \frac{u_{g-e} + u_{g-0}}{2} \right) \delta_2 H \times \left( \frac{u_{g-e} + u_{g-0}}{2} \right) + \rho_g \left( \frac{u_{g-0} + u_{g-w}}{2} \right) \delta_2 H \times \left( \frac{u_{g-0} + u_{g-w}}{2} \right) - \rho_g \left( \frac{u_{g-n} + u_{g-0}}{2} \right) \delta_2 W \times \left( \frac{v_{g-n} + v_{g-0}}{2} \right) + \rho_g \left( \frac{u_{g-0} + u_{g-s}}{2} \right) \delta_2 W \times \left( \frac{v_{g-0} + v_{g-s}}{2} \right)$

$\Rightarrow -(P_2 - P_1)_x + \frac{\int_A \tau_{i,x} dA}{\delta_1 H} - \frac{\int_A \tau_{wf,x} dA}{\delta_1 H} = -\rho_f \left( \frac{u_{f-e} + u_{f-0}}{2} \right)^2 + \rho_f \left( \frac{u_{f-0} + u_{f-w}}{2} \right)^2 - \rho_f \left( \frac{u_{f-n} + u_{f-0}}{2} \right) \frac{W}{H} \times \left( \frac{v_{f-n} + v_{f-0}}{2} \right) + \rho_f \left( \frac{u_{f-0} + u_{f-s}}{2} \right) \frac{W}{H} \times \left( \frac{v_{f-0} + v_{f-s}}{2} \right)$

$-) -(P_2 - P_1)_x - \frac{\int_A \tau_{i,x} dA}{\delta_2 H} - \frac{\int_A \tau_{wg,x} dA}{\delta_2 H} = -\rho_g \left( \frac{u_{g-e} + u_{g-0}}{2} \right)^2 + \rho_g \left( \frac{u_{g-0} + u_{g-w}}{2} \right)^2 - \rho_g \left( \frac{u_{g-n} + u_{g-0}}{2} \right) \frac{W}{H} \times \left( \frac{v_{g-n} + v_{g-0}}{2} \right) + \rho_g \left( \frac{u_{g-0} + u_{g-s}}{2} \right) \frac{W}{H} \times \left( \frac{v_{g-0} + v_{g-s}}{2} \right)$

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$$\frac{\int_A \tau_{i,x} dA}{\delta_1 H} - \frac{\int_A \tau_{wf,x} dA}{\delta_1 H} + \frac{\int_A \tau_{i,x} dA}{\delta_2 H} + \frac{\int_A \tau_{wg,x} dA}{\delta_2 H} = -\rho_f \left( \frac{u_{f-e} + u_{f-0}}{2} \right)^2 + \rho_f \left( \frac{u_{f-0} + u_{f-w}}{2} \right)^2 - \rho_f \left( \frac{u_{f-n} + u_{f-0}}{2} \right) \frac{W}{H} \times \left( \frac{v_{f-n} + v_{f-0}}{2} \right) + \rho_f \left( \frac{u_{f-0} + u_{f-s}}{2} \right) \frac{W}{H} \times \left( \frac{v_{f-0} + v_{f-s}}{2} \right) + \rho_g \left( \frac{u_{g-e} + u_{g-0}}{2} \right)^2 - \rho_g \left( \frac{u_{g-0} + u_{g-w}}{2} \right)^2 + \rho_g \left( \frac{u_{g-n} + u_{g-0}}{2} \right) \frac{W}{H} \times \left( \frac{v_{g-n} + v_{g-0}}{2} \right) - \rho_g \left( \frac{u_{g-0} + u_{g-s}}{2} \right) \frac{W}{H} \times \left( \frac{v_{g-0} + v_{g-s}}{2} \right)$$

# Momentum Conservation Equations

## Y-direction force balance

Liquid film  $-(P_2 - P_1)_y \delta_1 W - \int_A \tau_{i,y} dA - \int_A \tau_{wf,y} dA + \rho_f g \delta_1 H W = -\rho_f \left( \frac{v_{f-n} + v_{f-0}}{2} \right) \delta_1 W \times \left( \frac{v_{f-n} + v_{f-0}}{2} \right) + \rho_f \left( \frac{v_{f-0} + v_{f-s}}{2} \right) \delta_1 W \times \left( \frac{v_{f-0} + v_{f-s}}{2} \right) - \rho_f \left( \frac{v_{f-e} + v_{f-0}}{2} \right) \delta_1 H \times \left( \frac{u_{f-e} + u_{f-0}}{2} \right) + \rho_f \left( \frac{v_{f-0} + v_{f-w}}{2} \right) \delta_1 H \times \left( \frac{u_{f-0} + u_{f-w}}{2} \right)$

Air  $-(P_2 - P_1)_y \delta_2 W + \int_A \tau_{i,y} dA - \int_A \tau_{wg,y} dA + \rho_g g \delta_2 H W = -\rho_g \left( \frac{v_{g-n} + v_{g-0}}{2} \right) \delta_2 W \times \left( \frac{v_{g-n} + v_{g-0}}{2} \right) + \rho_g \left( \frac{v_{g-0} + v_{g-s}}{2} \right) \delta_2 W \times \left( \frac{v_{g-0} + v_{g-s}}{2} \right) - \rho_g \left( \frac{v_{g-e} + v_{g-0}}{2} \right) \delta_2 H \times \left( \frac{u_{g-e} + u_{g-0}}{2} \right) + \rho_g \left( \frac{v_{g-0} + v_{g-w}}{2} \right) \delta_2 H \times \left( \frac{u_{g-0} + u_{g-w}}{2} \right)$

$\rightarrow -(P_2 - P_1)_y - \frac{\int_A \tau_{i,y} dA}{\delta_1 W} - \frac{\int_A \tau_{wf,y} dA}{\delta_1 W} + \rho_f g H = -\rho_f \left( \frac{v_{f-n} + v_{f-0}}{2} \right)^2 + \rho_f \left( \frac{v_{f-0} + v_{f-s}}{2} \right)^2 - \rho_f \left( \frac{v_{f-e} + v_{f-0}}{2} \right) \frac{H}{W} \times \left( \frac{u_{f-e} + u_{f-0}}{2} \right) + \rho_f \left( \frac{v_{f-0} + v_{f-w}}{2} \right) \frac{H}{W} \times \left( \frac{u_{f-0} + u_{f-w}}{2} \right)$

$-) -(P_2 - P_1)_y + \frac{\int_A \tau_{i,y} dA}{\delta_2 W} + \rho_g g H = -\rho_g \left( \frac{v_{g-n} + v_{g-0}}{2} \right)^2 + \rho_g \left( \frac{v_{g-0} + v_{g-s}}{2} \right)^2 - \rho_g \left( \frac{v_{g-e} + v_{g-0}}{2} \right) \frac{H}{W} \times \left( \frac{u_{g-e} + u_{g-0}}{2} \right) + \rho_g \left( \frac{v_{g-0} + v_{g-w}}{2} \right) \frac{H}{W} \times \left( \frac{u_{g-0} + u_{g-w}}{2} \right)$

$-\frac{\int_A \tau_{i,y} dA}{\delta_1 W} - \frac{\int_A \tau_{wf,y} dA}{\delta_1 W} - \frac{\int_A \tau_{i,y} dA}{\delta_2 W} + \rho_f g H - \rho_g g H = -\rho_f \left( \frac{v_{f-n} + v_{f-0}}{2} \right)^2 + \rho_f \left( \frac{v_{f-0} + v_{f-s}}{2} \right)^2 - \rho_f \left( \frac{v_{f-e} + v_{f-0}}{2} \right) \frac{H}{W} \times \left( \frac{u_{f-e} + u_{f-0}}{2} \right) + \rho_f \left( \frac{v_{f-0} + v_{f-w}}{2} \right) \frac{H}{W} \times \left( \frac{u_{f-0} + u_{f-w}}{2} \right) + \rho_g \left( \frac{v_{g-n} + v_{g-0}}{2} \right)^2 - \rho_g \left( \frac{v_{g-0} + v_{g-s}}{2} \right)^2 + \rho_g \left( \frac{v_{g-e} + v_{g-0}}{2} \right) \frac{H}{W} \times \left( \frac{u_{g-e} + u_{g-0}}{2} \right) - \rho_g \left( \frac{v_{g-0} + v_{g-w}}{2} \right) \frac{H}{W} \times \left( \frac{u_{g-0} + u_{g-w}}{2} \right)$

# Momentum Conservation Equations

$$\frac{\int_A \tau_{i,x} dA}{\delta_1 H} - \frac{\int_A \tau_{wf,x} dA}{\delta_1 H} + \frac{\int_A \tau_{i,x} dA}{\delta_2 H} + \frac{\int_A \tau_{wg,x} dA}{\delta_2 H} = -\rho_f \left( \frac{u_{f-e} + u_{f-0}}{2} \right)^2 + \rho_f \left( \frac{u_{f-0} + u_{f-w}}{2} \right)^2 - \rho_f \left( \frac{u_{f-n} + u_{f-0}}{2} \right) \frac{W}{H} \times \left( \frac{v_{f-n} + v_{f-0}}{2} \right) + \rho_f \left( \frac{u_{f-0} + u_{f-s}}{2} \right) \frac{W}{H} \times \left( \frac{v_{f-0} + v_{f-s}}{2} \right) + \rho_g \left( \frac{u_{g-e} + u_{g-0}}{2} \right)^2 - \rho_g \left( \frac{u_{g-0} + u_{g-w}}{2} \right)^2 + \rho_g \left( \frac{u_{g-n} + u_{g-0}}{2} \right) \frac{W}{H} \times \left( \frac{v_{g-n} + v_{g-0}}{2} \right) - \rho_g \left( \frac{u_{g-0} + u_{g-s}}{2} \right) \frac{W}{H} \times \left( \frac{v_{g-0} + v_{g-s}}{2} \right)$$

$$\frac{\int_A \tau_{i,y} dA}{\delta_1 W} - \frac{\int_A \tau_{wf,y} dA}{\delta_1 W} - \frac{\int_A \tau_{i,y} dA}{\delta_2 W} + \rho_f g H - \rho_g g H = -\rho_f \left( \frac{v_{f-n} + v_{f-0}}{2} \right)^2 + \rho_f \left( \frac{v_{f-0} + v_{f-s}}{2} \right)^2 - \rho_f \left( \frac{v_{f-e} + v_{f-0}}{2} \right) \frac{H}{W} \times \left( \frac{u_{f-e} + u_{f-0}}{2} \right) + \rho_f \left( \frac{v_{f-0} + v_{f-w}}{2} \right) \frac{H}{W} \times \left( \frac{u_{f-0} + u_{f-w}}{2} \right) + \rho_g \left( \frac{v_{g-n} + v_{g-0}}{2} \right)^2 - \rho_g \left( \frac{v_{g-0} + v_{g-s}}{2} \right)^2 + \rho_g \left( \frac{v_{g-e} + v_{g-0}}{2} \right) \frac{H}{W} \times \left( \frac{u_{g-e} + u_{g-0}}{2} \right) - \rho_g \left( \frac{v_{g-0} + v_{g-w}}{2} \right) \frac{H}{W} \times \left( \frac{u_{g-0} + u_{g-w}}{2} \right)$$

$$\int_A \tau_{i,x} dA = \int_A \frac{1}{2} \rho_g f_i \overline{U}_g - \overline{U}_f \left| (U_g - U_f) \right| dA = \int_A \frac{1}{2} \rho_g f_i (u_g(x) - u_f(x)) \sqrt{(u_g(x) - u_f(x))^2 + v_f^2(y)} dA$$

$$\int_A \tau_{wf,x} dA = \int_A \frac{1}{2} \rho_f f_{wf} u_f(x) \sqrt{u_f^2(x) + v_f^2(y)} dA$$

$$\int_A \tau_{wg,x} dA = \int_A \frac{1}{2} \rho_g f_{wg} u_g^2(x) dA$$

Blasius' equation

$$f_{wg} = 0.79 \text{Re}^{-1/4}$$

$$\int_A \tau_{i,y} dA = \int_A \frac{1}{2} \rho_g f_i \overline{U}_g - \overline{U}_f \left| (U_g - U_f) \right| dA = \int_A \frac{1}{2} \rho_g f_i v_f(y) \sqrt{(u_g(x) - u_f(x))^2 + v_f^2(y)} dA$$

$$\int_A \tau_{wf,y} dA = \int_A \frac{1}{2} \rho_f f_{wf} v_f(y) \sqrt{u_f^2(x) + v_f^2(y)} dA$$

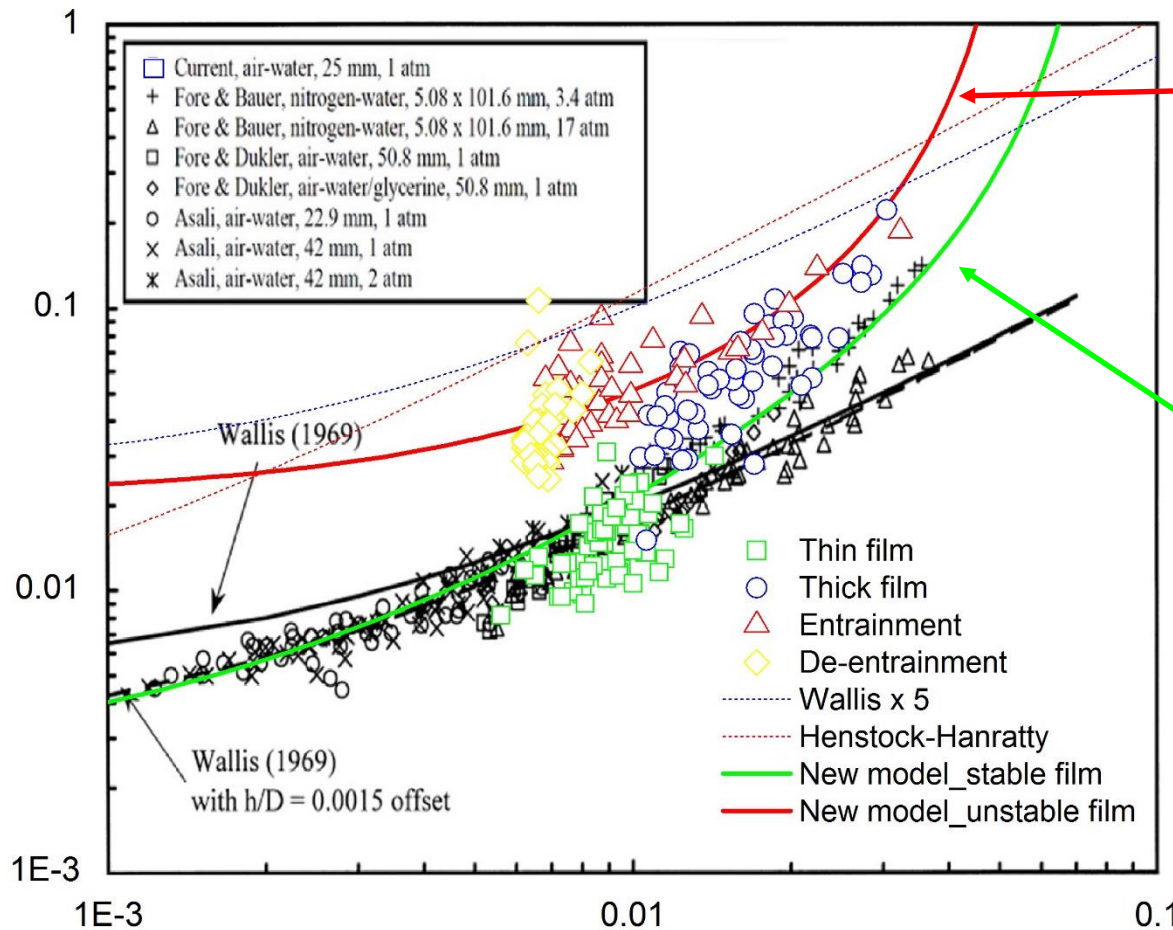
Assume that...

- ① Ignore lateral air flow acceleration
- ② Linear velocity profile
- ③ Constant friction coefficient
- ④ Gas wall friction factor follows Blasius' Eq.

$$\begin{matrix} \rightarrow & \begin{pmatrix} a_1 & a_2 \\ a_3 & a_4 \end{pmatrix} \begin{pmatrix} f_i \\ f_w \end{pmatrix} = \begin{pmatrix} b_1 \\ b_2 \end{pmatrix} \end{matrix}$$



# Comparison with Previous Studies



$$f_i = \frac{1}{4} \left[ 1.879 - 32.928 \left( \frac{h}{D} - 0.0015 \right) \right]^{-2} - 0.046$$

Unstable film flow  
(entrainment, de-entrainment)

$$f_i = \frac{1}{4} \left[ 1.879 - 22.104 \left( \frac{h}{D} - 0.0015 \right) \right]^{-2} - 0.066$$

Stable film flow  
(thin film, thick film w/o entrainment)

Flow pattern	Avg. error (%)	Std. error (%)
Thin film	-36	32
Thick film	32	23
Entrainment	-2	24
De-entrainment	-15	26