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Verification of Heat Exchanger Design Code KAIST_HXD by Experiment

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CONTENTS

I

Introduction

II

Experimental Facility

III

Heat Exchanger Design Code

IV

Summary & Further Works

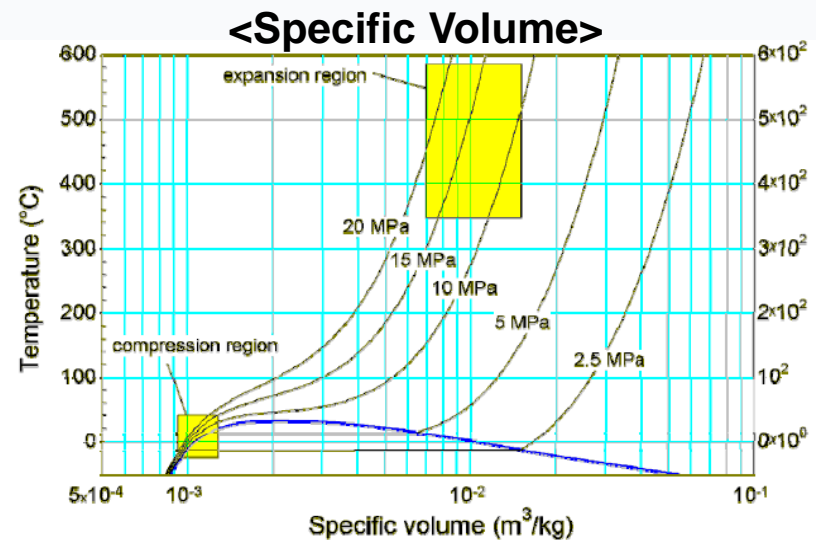
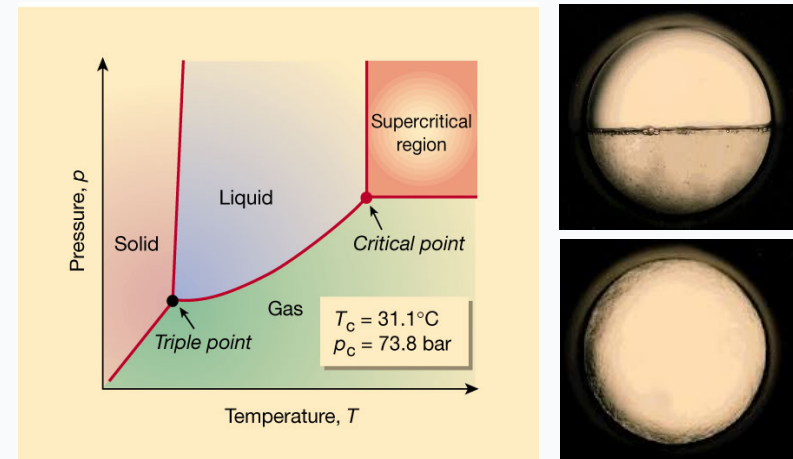
V

References

1. Introduction

(1) What is Supercritical CO₂ & Why CO₂ ?

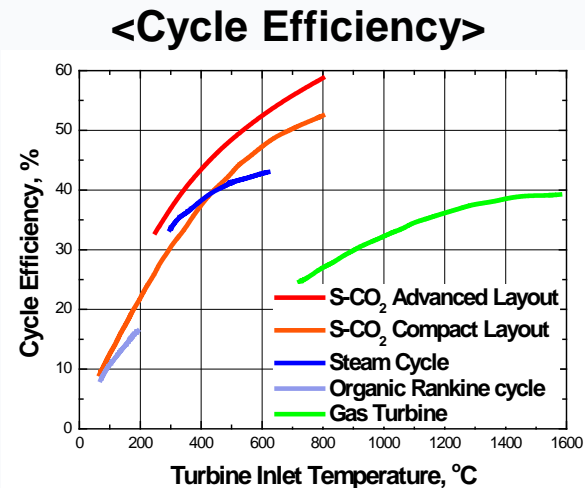
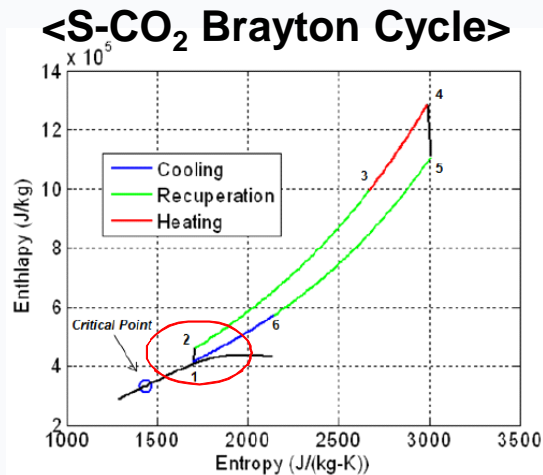
Substance ^{[3][4]}	Critical temperature	Critical pressure
Helium	-267.96 °C (5.19 K)	2.24 atm (227 kPa)
Hydrogen	-239.95 °C (33.20 K)	12.8 atm (1,300 kPa)
Neon	-228.75 °C (44.40 K)	27.2 atm (2,760 kPa)
CH ₄	-82.3 °C (190.8 K)	45.79 atm (4,640 kPa)
Nitrogen	-146.9 °C (126.2 K)	33.5 atm (3,390 kPa)
Fluorine	-128.85 °C (144.30 K)	51.5 atm (5,220 kPa)
Argon	-122.4 °C (150.7 K)	48.1 atm (4,870 kPa)
Oxygen	-118.6 °C (154.5 K)	49.8 atm (5,050 kPa)
Krypton	-63.8 °C (209.3 K)	54.3 atm (5,500 kPa)
Xenon	16.6 °C (289.8 K)	57.6 atm (5,840 kPa)
CO ₂	31.04 °C (304.19 K)	72.8 atm (7,380 kPa)
N ₂ O	36.4 °C (309.5 K)	71.5 atm (7,240 kPa)
Ammonia ^[5]	132.4 °C (405.5 K)	111.3 atm (11,280 kPa)
Chlorine	143.8 °C (417.0 K)	76.0 atm (7,700 kPa)
Bromine	310.8 °C (584.0 K)	102 atm (10,300 kPa)
Water ^{[6][7]}	373.946 °C (647.096 K)	217.7 atm (22,060 kPa)
H ₂ SO ₄	654 °C (927 K)	45.4 atm (4,600 kPa)
Sulfur	1,040.85 °C (1,314.00 K)	207 atm (21,000 kPa)
Mercury	1,476.9 °C (1,750.1 K)	1,720 atm (174,000 kPa)
Caesium	1,664.85 °C (1,938.00 K)	94 atm (9,500 kPa)
Ethanol	241 °C	62.18 atm (63 bar, 6,300 kPa)
Lithium	2,950 °C (3,220 K)	652 atm (66,100 kPa)



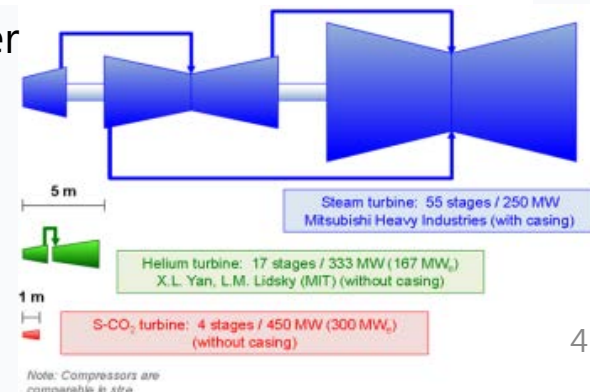
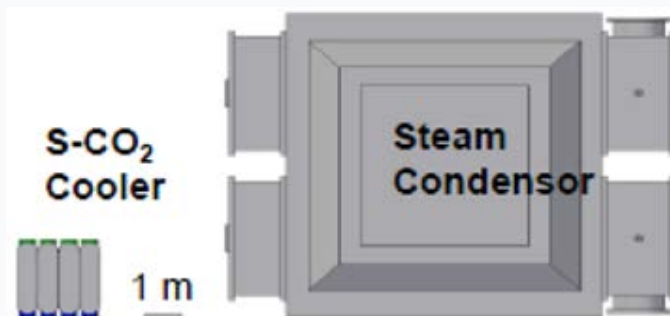
1. Introduction

(2) Supercritical CO₂ Brayton Cycle

- ❖ **Maximizing advantages** from Steam Rankine and Brayton cycle
 - Steam Rankine cycle → Small pumping work
 - Gas Brayton cycle → High efficiency in High T.I.T
- ❖ **Simple layout** is sufficient to achieve **high efficiency**



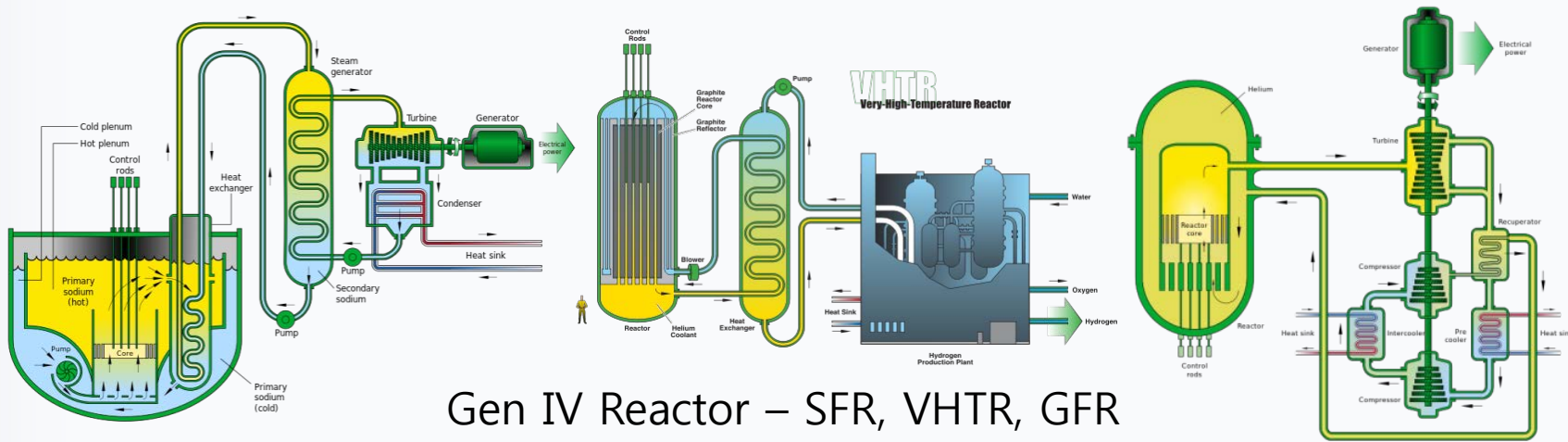
- ❖ **Compact** Turbomachinery & Heat exchanger



1. Introduction

(3) Supercritical CO₂ Brayton cycle application area

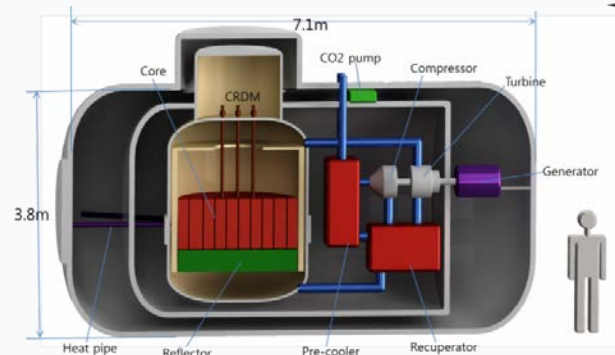
- ❖ Generation IV reactor
 - Sodium cooled fast reactor
 - Very high temperature reactor
 - Gas cooled fast reactor



Gen IV Reactor – SFR, VHTR, GFR

Reducing initial investment cost & Modularize

- ❖ Small distributed power
- ❖ Ship propulsion

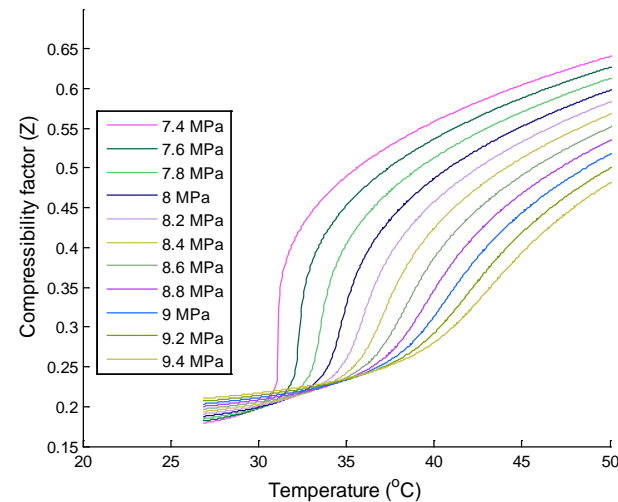
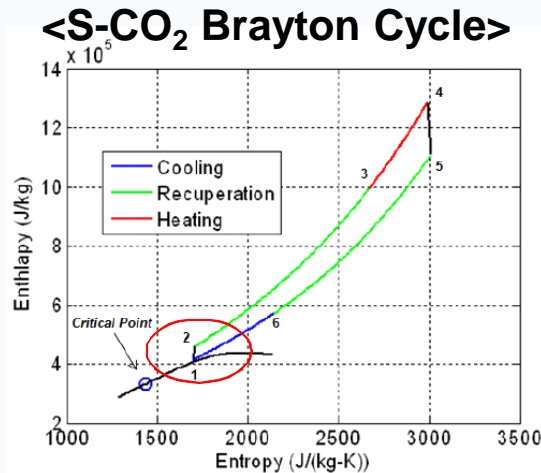


KAIST - MMR

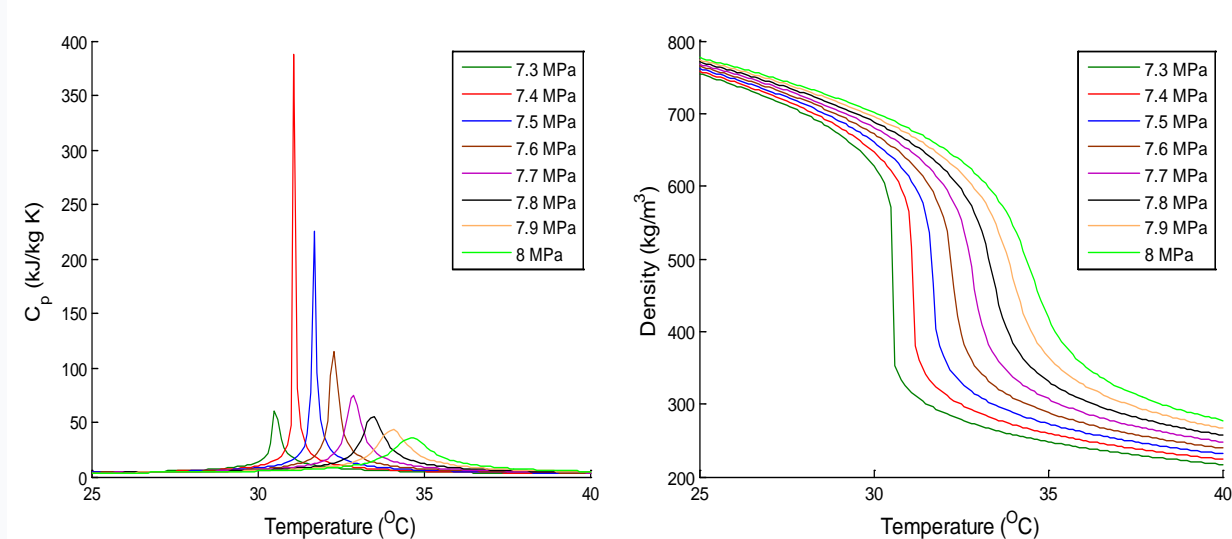
1. Introduction

(4) Supercritical CO₂ cycle characteristics

- ❖ Compressor inlet condition control is important.



- ❖ Dramatic property change of S-CO₂ near the critical point (31°C, 7.4MPa)



1. Introduction

(5) Research objectives

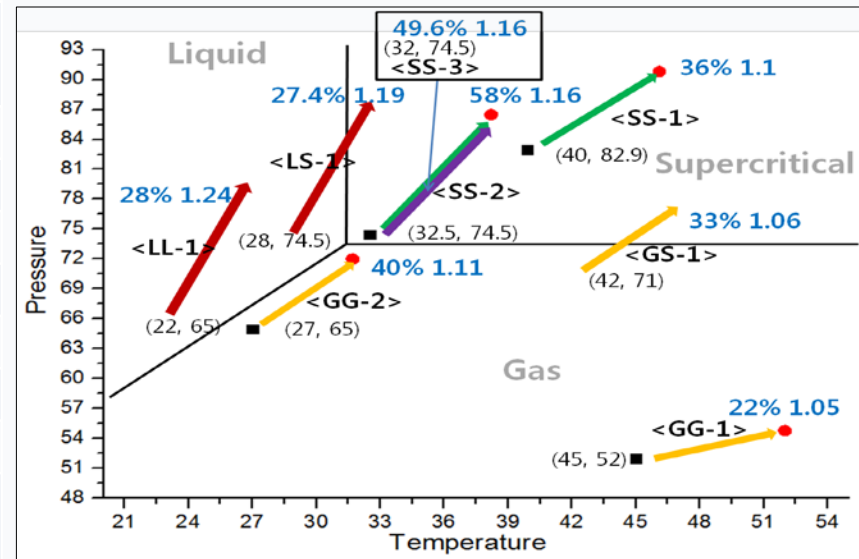
- ❖ **Precooler** (emitting the waste heat) temperature control is important.
 - Compressor inlet condition is solely dependent on the precooler performance.
- ❖ Cycle minimum temperature affects more to the cycle efficiency rather than cycle operating maximum temperature.
- ❖ **Difficulty** in computational analysis or numerical design approach due to the dramatic property change of S-CO₂ near the critical point .
- ❖ Experiment and real operation experiences are necessary to support.
- ❖ **Proper Precooler design!!**
 - **Heat exchanger design code, validation, Performance test**

2. Experimental Facility

➤ S-CO₂PE [Supercritical CO₂ Pressurizing Experiment]

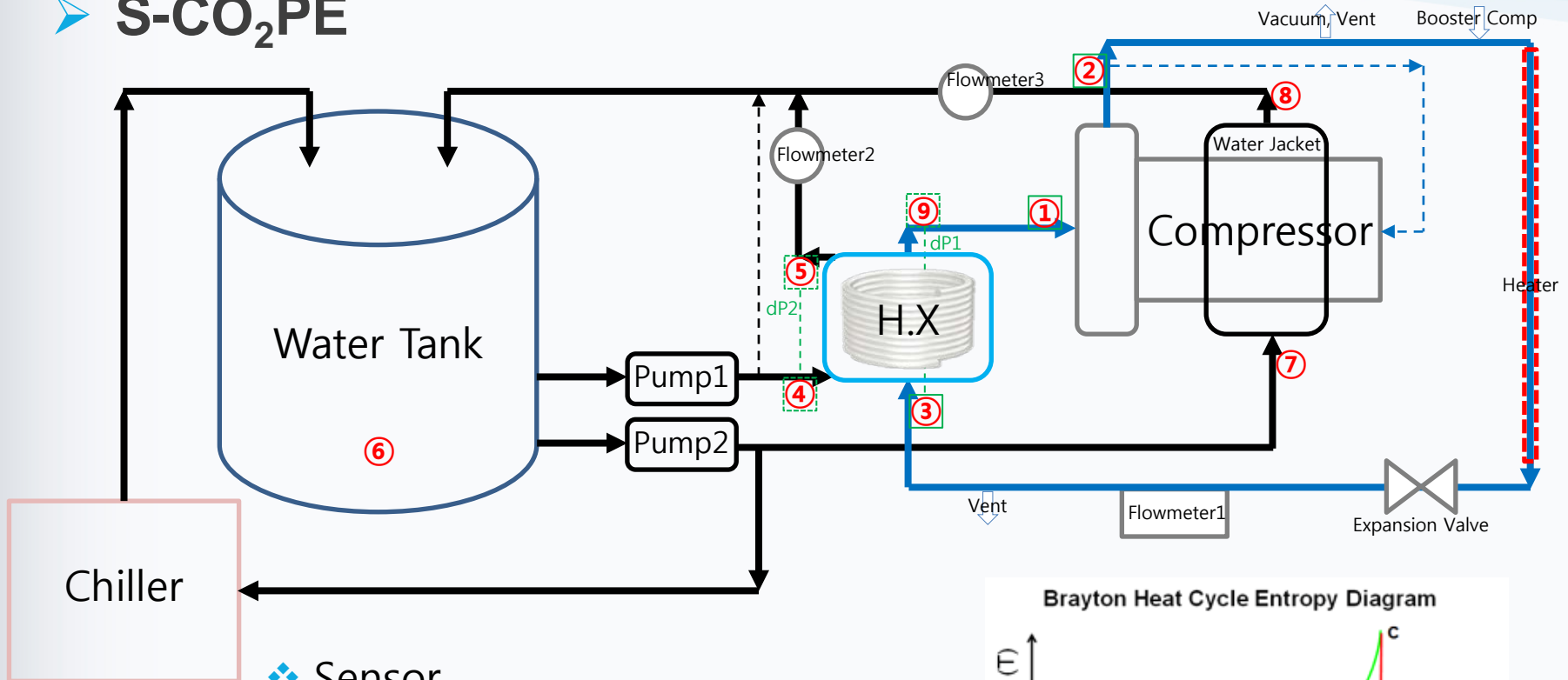


Compressor type		Seal-free canned motor pump
Compressor power [kW]		26
Mass flow [kg/s]		2.78
Compressor pressure ratio		1.2
Maximum RPM		4620
Inlet Condition	Pressure	7.56 (MPa)
	Temperature	32 (°C)
Electrical heater [kW]		0.5
Precooler type		Spiral tube heat exchanger



2. Experimental Facility

➤ S-CO₂PE



❖ Sensor

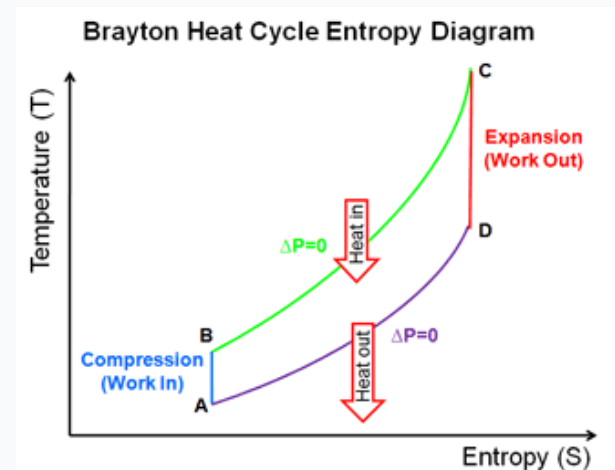
RTD sensor : 9

Pressure transmitter : 3

Differential pressure gauge : 2

Mass flowmeter(CO₂) : 1

Flowmeter(water) : 2



2. Experimental Facility

➤ Sensor Accuracy

Sensor type	Range	Accuracy
RTD	0 ~ 100 °C	± 0.2°C
Pressure transmitter	0 ~ 120 bar	± 0.05%
Differential pressure gauge (CO ₂)	0 ~ 250 kPa	± 0.065%
Differential pressure gauge (water)	0.5 ~ 100 kPa	± 0.04%
Mass flow meter (CO ₂)	0 ~ 5 kg/s	± 0.16%
Flow meter (water)	0 ~ 3.33 kg/s	± 0.5%

- ❖ To decrease the measurement error
 - The RTD sensors were immersed in thermostat for calibration.
 - The pressure gauges and differential gauges were tested with calibrator.

2. Experimental Facility

➤ STHE

- ❖ Spiral Tube Heat Exchanger
- ❖ SENTRY EQUIPMENT CORP.

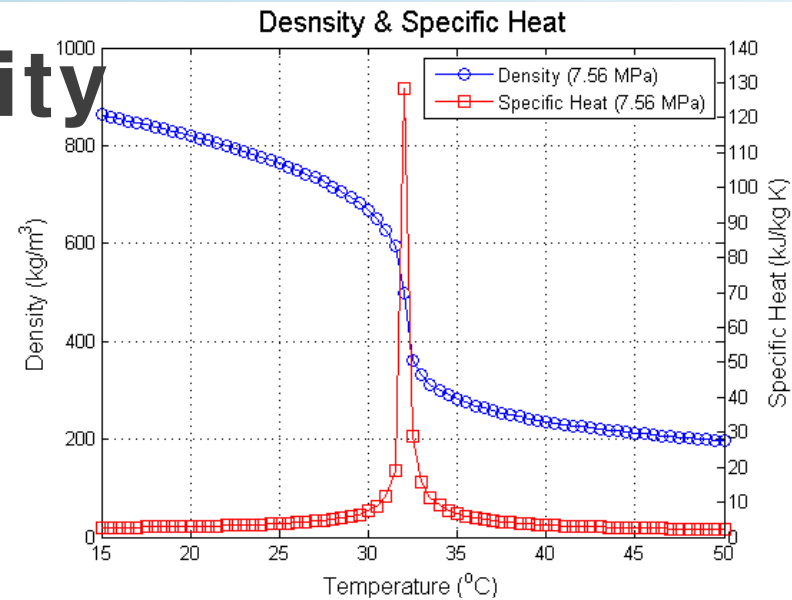


- Specification data - LMTD method

	Tube side	Shell side
Fluid type	CO ₂	Water
Mass flow rate [kg/s]	2.78	1.052
Inlet Temp. [°C]	32.06	7
Outlet Temp. [°C]	32	12.3
Inlet pressure [MPa]	7.56	0.45
Pressure drop [kPa]	48	13
Heat transfer area [m ²]	0.635	
Heat load [kW]	23.4	
Overall Heat transfer coefficient [W/m ² K]	1656.4	
Log mean temperature difference [°C]	22.27	
Volume [m ³]	0.01987	
Diameter [m]	0.324	
Length [m]	0.241	

2. Experimental Facility

- STHE Experiment data
- LMTD method : **constant C_p**



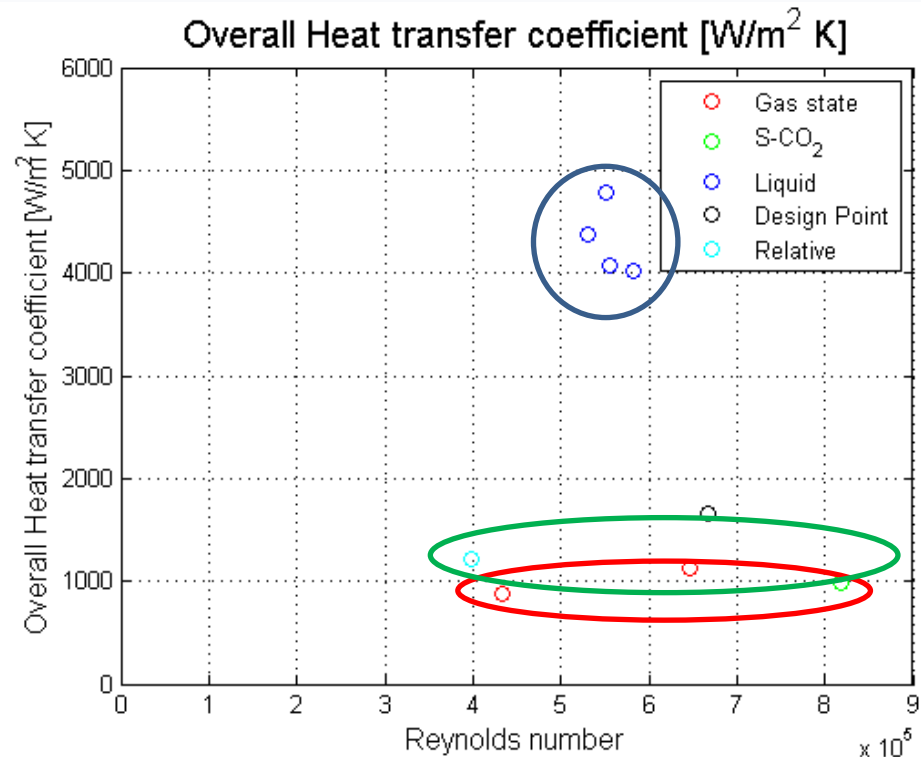
	Tube side	Shell side	Tube side	Shell side
Fluid type	CO ₂	Water	CO ₂	Water
Mass flow rate [kg/s]	2.78	1.052	1.016	0.078
Inlet Temp. [°C]	32.06	7	36.605	11.241
Outlet Temp. [°C]	32	12.3	34.662	33.869
Inlet pressure [MPa]	7.56	0.45	7.425	0.45
Pressure drop [kPa]	48	13	47.58	0.03
Heat transfer area [m ²]	0.635		-	
Heat load [kW]	23.4		7.4	
Overall Heat transfer coefficient [W/m ² K]	1656.4		1222.7	
Log mean temperature difference [°C]	22.27		9.6337	

2. Experimental Facility

➤ STHE Experiment data

- ❖ Overall heat transfer coefficient :
 - Liquid > Supercritical state > gas state
- ❖ High Reynolds number
 - Low viscosity
 - Inner tube diameter : 8mm

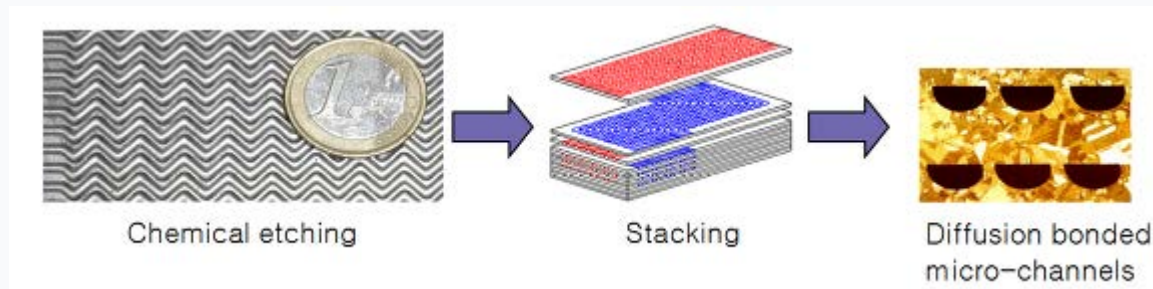
Case	Mass flow rate [kg/s]	Inlet Condit' [MPa / °C]	Phase	Density [kg/m ³]	Re	U [W/m ² K]
1	0.93	5.42 / 50.3	Gas	116.71	434301	881.4
2	1.55	7.43 / 46.3	Gas	201.09	647222	1126.1
3	2.49	8.66 / 42.5	Supercritical	330.22	819131	985.1
4	4.28	6.95 / 23.9	Liquid	758.54	550405	4779.2
5	4.34	7.6 / 23.9	Liquid	778.99	531180	4369.9
6	4.34	7.16 / 24.3	Liquid	760.18	555834	4082.8
7	3.92	7.94 / 30.1	Liquid	695.97	582294	4026.1



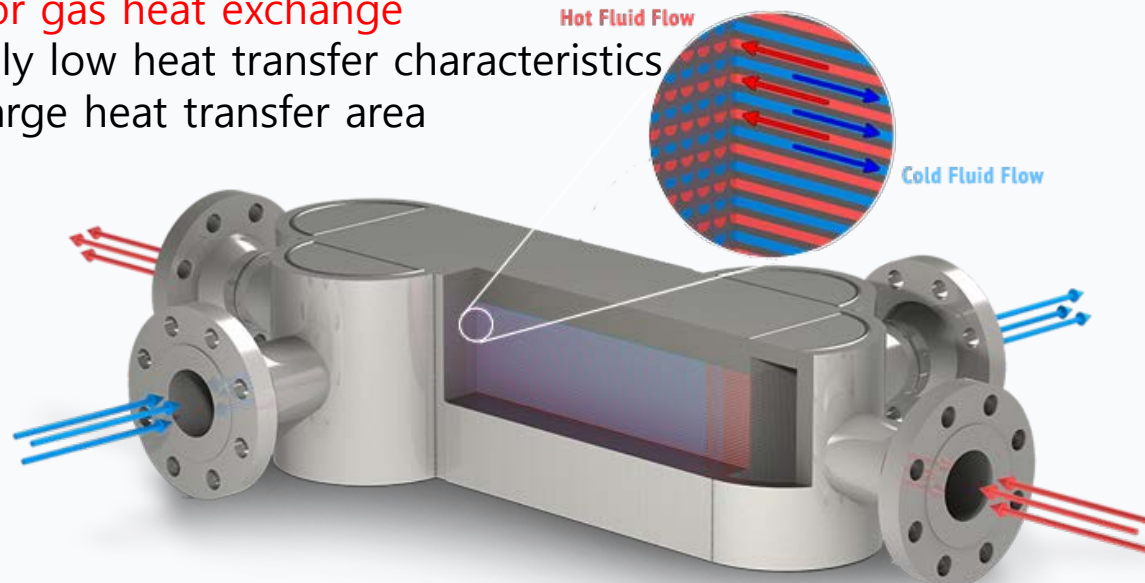
3. Heat exchanger design code KAIST_HXD

3. Heat Exchanger Design Code

➤ PCHE (Printed Circuit Heat Exchanger)



- ❖ **High Temperature, High Pressure** (S-CO₂ power cycle application)
- ❖ **High Compactness**[m²/m³] Heat exchanger
- ❖ Available for **gas heat exchange**
 - Relatively low heat transfer characteristics
 - Need large heat transfer area

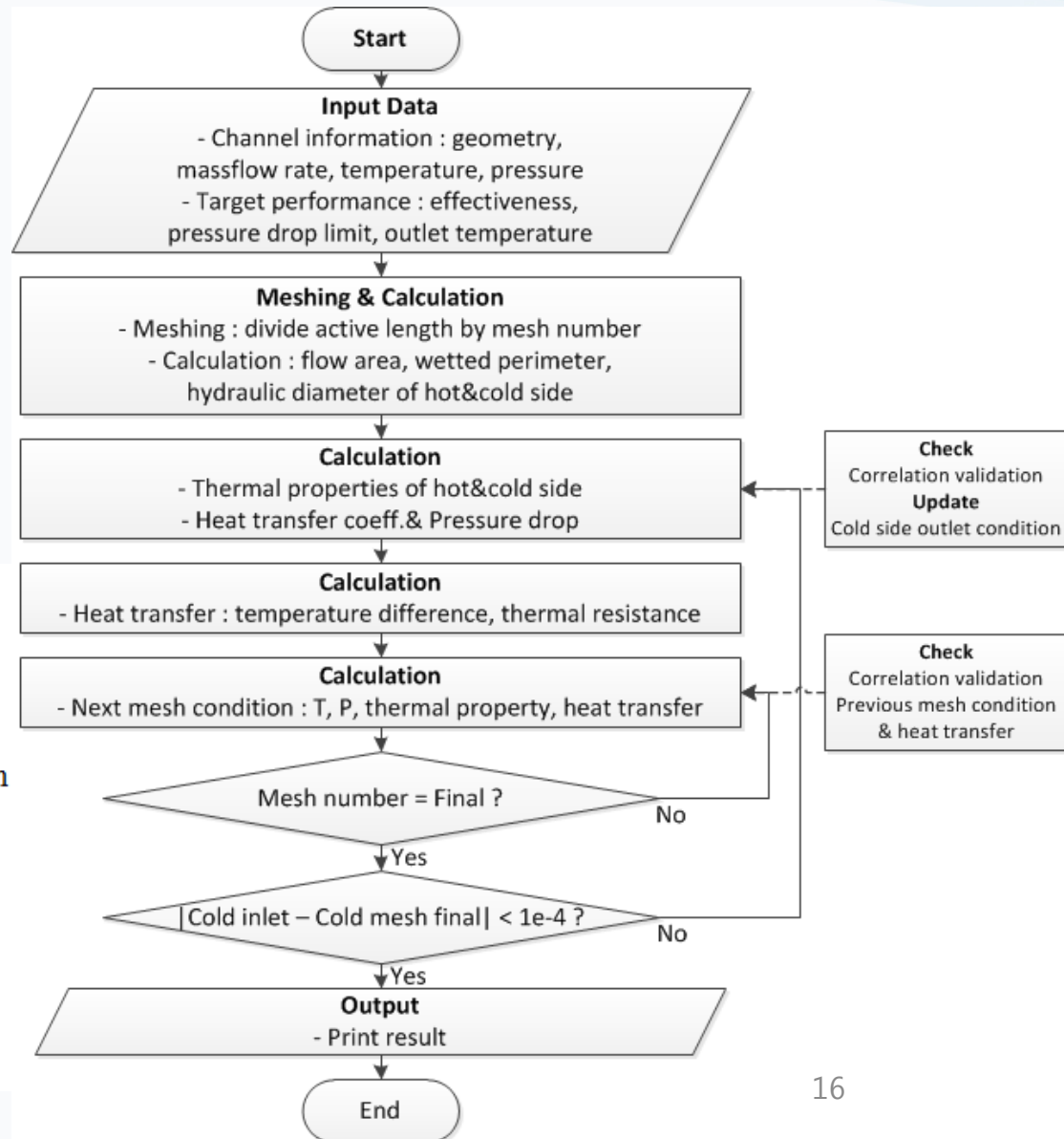
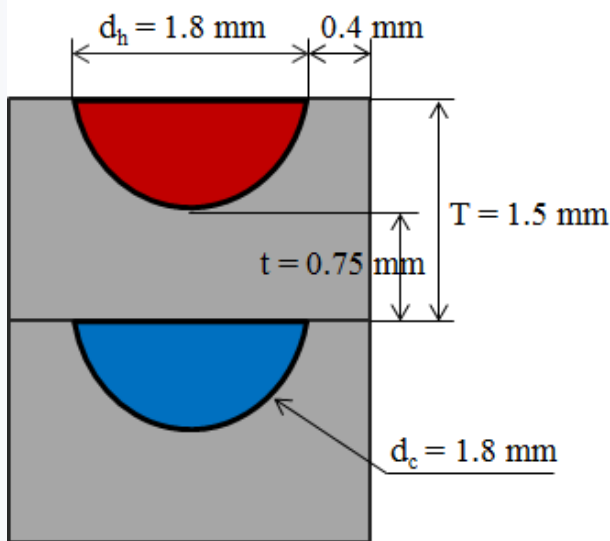


PCHE from Vacuum Process Engineering

3. Heat Exchanger Design Code

➤ KAIST_HXD

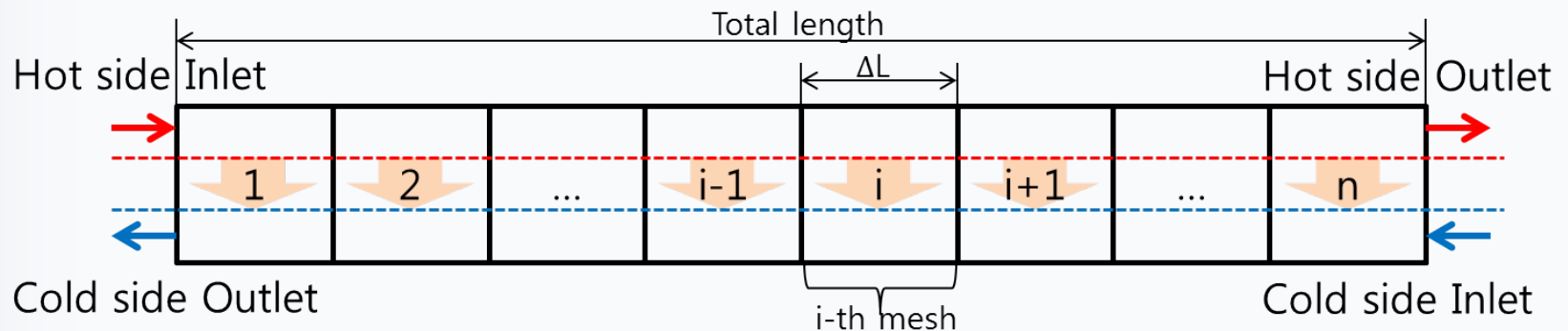
- ❖ MATLAB based code
 - Performance
 - Pressure drop
- ❖ PCHE core design
 - Flow type :
 - Counter-current
 - Cross flow
 - Exclude header



3. Heat Exchanger Design Code

➤ KAIST_HXD

- ❖ Due to the repetitive channel geometry of PCHE, the overall computation can be simply interpreted from a set of **representative hot and cold unit channel**



- ❖ Heat transfer

$$Q = U A \Delta T = \frac{1}{R_{conv.Hot} + R_{cond} + R_{conv.Cold}} A \Delta T = \frac{1}{\frac{1}{h_{Hot}} + \frac{t}{k_{cond}} + \frac{1}{h_{Cold}}} A \Delta T$$

- ❖ Pressure drop

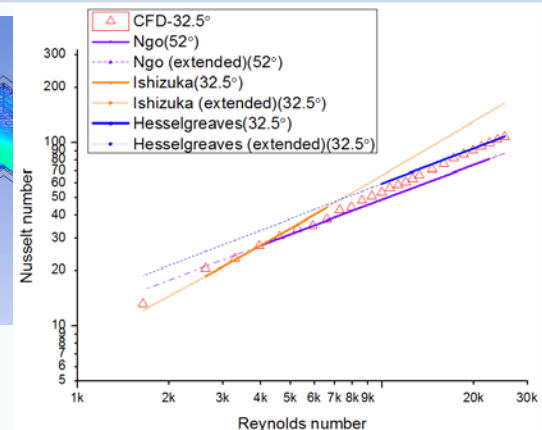
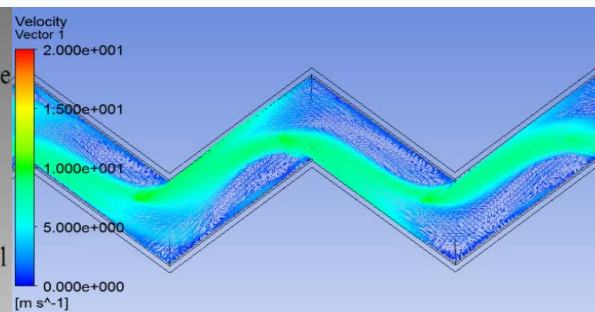
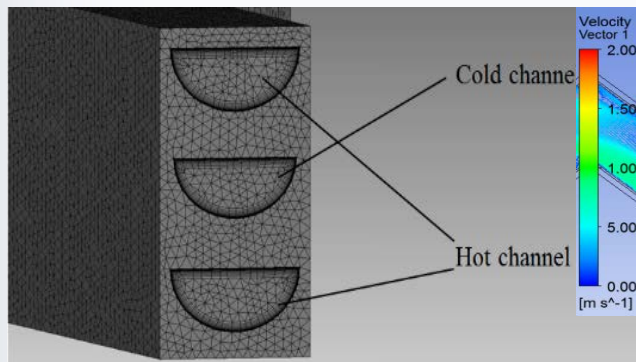
$$\Delta P = 4f \frac{l}{D} \frac{\rho V^2}{2}$$

3. Heat Exchanger Design Code

➤ KAIST_HXD

- ❖ Heat transfer coefficient (Nusselt number) & Friction factor
 - CFD analysis (ANSYS CFX) [Seung Gu Kim et al.]
 - Experimental study of Ishizuka (2400 < Re < 6000)

Fin angle	32.5°
Diameter	1.9 mm
Reynolds number	2,000 < Re < 58,000
Nusselt number	$Nu = (0.02925 \pm 0.00153)Re^{0.8138 \pm 0.00501}$ $R^2 = 0.99904$
Friction factor	$f = (0.25150 \pm 0.00969)Re^{-0.20315 \pm 0.00414}$ $R^2 = 0.98341$



4. Summary & Further works

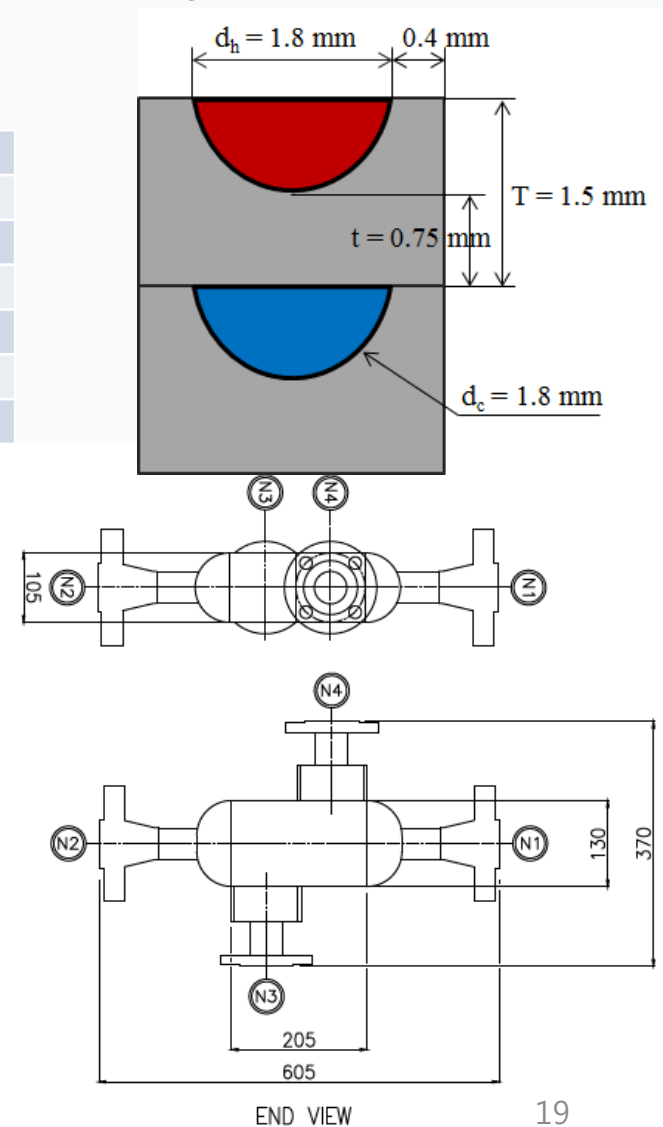
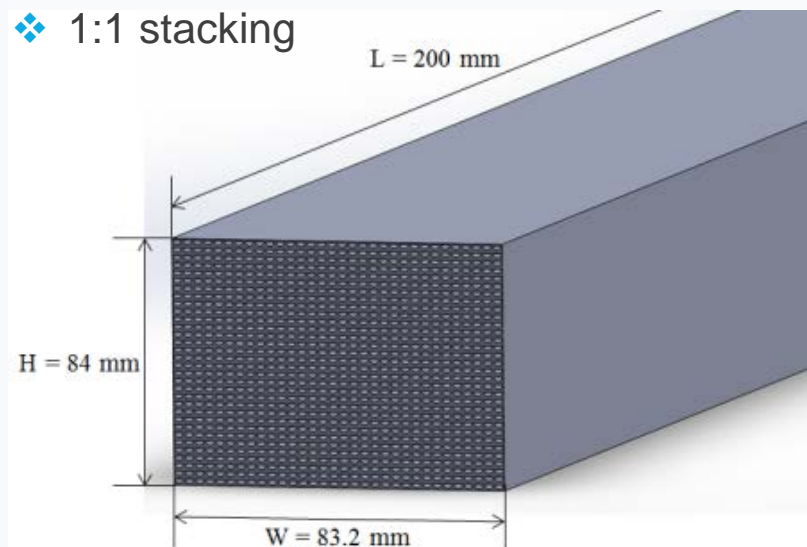
➤ Designed PCHE for S-CO₂PE facility

❖ Specification

Material	SS316L	
Density [g/cm³]	7.9	
Thermal conductivity [W/m·K]	15.3	
Hot side	Maximum Pressure [MPa]	19.5
	Maximum Temperature [°C]	150
Cold side	Maximum Pressure [MPa]	1.3
	Maximum Temperature [°C]	150

❖ 56 row x 32 channel (896channels each)

❖ 1:1 stacking



4. Summary & Further works

➤ Further works

- ❖ PCHE Experiment for KAIST_HXD code Validation
- ❖ CFD analysis comparing with experiment result
- ❖ Various range experiment for equilibrium-state cycle condition
- ❖ Comparing with STHE heat exchanger, PCHE availability, characteristics.

5. References

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THANK YOU